Train Brakes

These are the notes of a presentation made by Dominic Wells specially for the locomotive crews of the Ffestiniog and Welsh Highland Railways in 2011.

For best viewing, set the size to show one whole page only, and use the "Page Down" button to move through the slides.

If in doubt about any of the information contained within this presentation, please contact the author via Boston Lodge Works.

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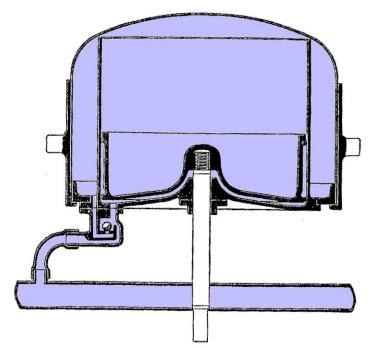
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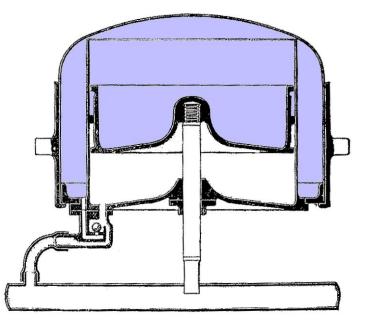
Brief history

- 1829 Rocket had no brake
 - first railway fatality
 - handbrakes introduced
- 1869 Westinghouse straight air brake
- 1889 Armagh disaster
 - Continuous braking compulsory for passenger trains
 - Automatic air brake & Automatic vacuum brake
- 1956 Introduction of distributor valve
- 1970s Proliferation of electro-pneumatic brakes

Introduction and principles

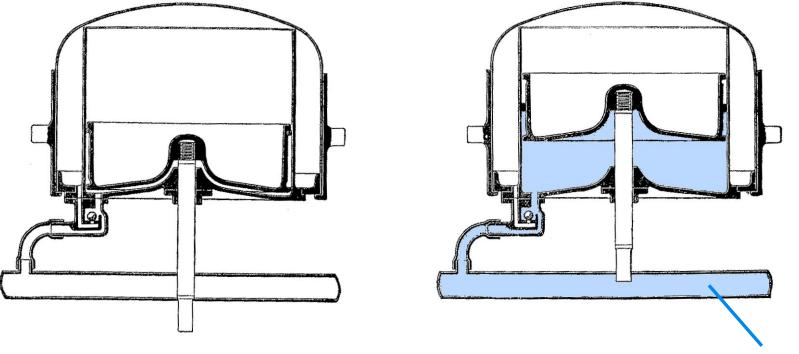
Here are the familiar diagrams of the vacuum brake. The purple areas represent a vacuum.





Brake released – piston falls under its own weight and moves the brake blocks away from the wheels. Brake applied – piston forced upwards when vacuum is destroyed in brake pipe. Brake blocks pulled onto the wheels.

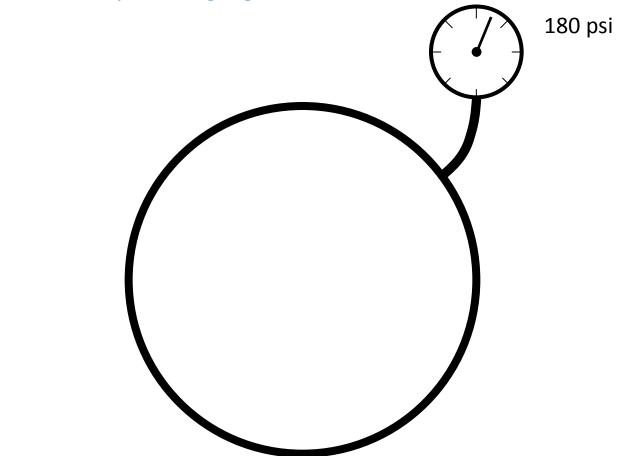
Or consider it another way. A "vacuum" is effectively nothing. Therefore, the vacuum brake system is actually a direct air brake using air at atmospheric pressure.

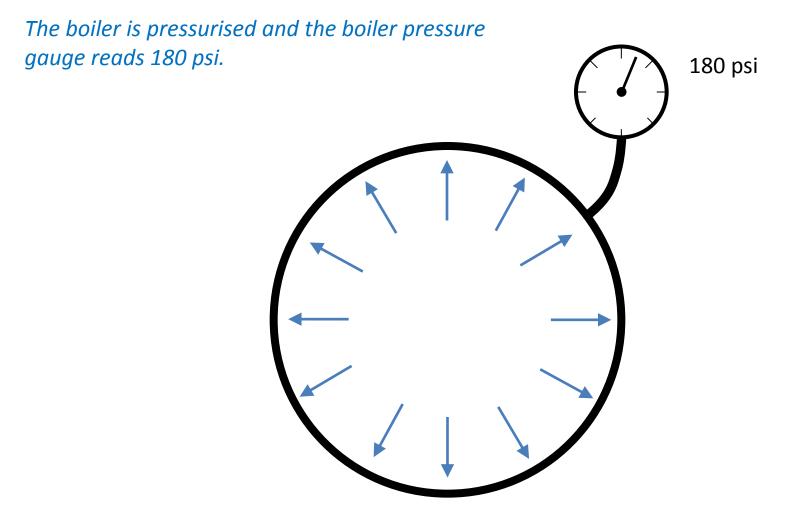


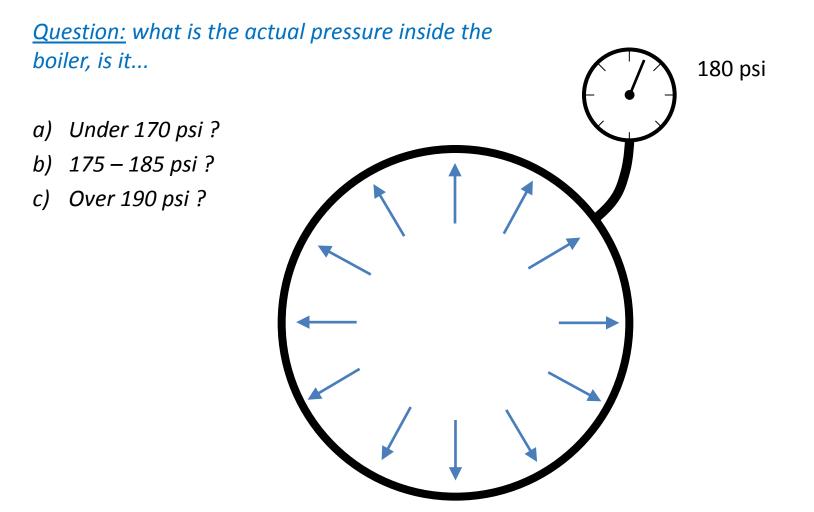
Air pressure

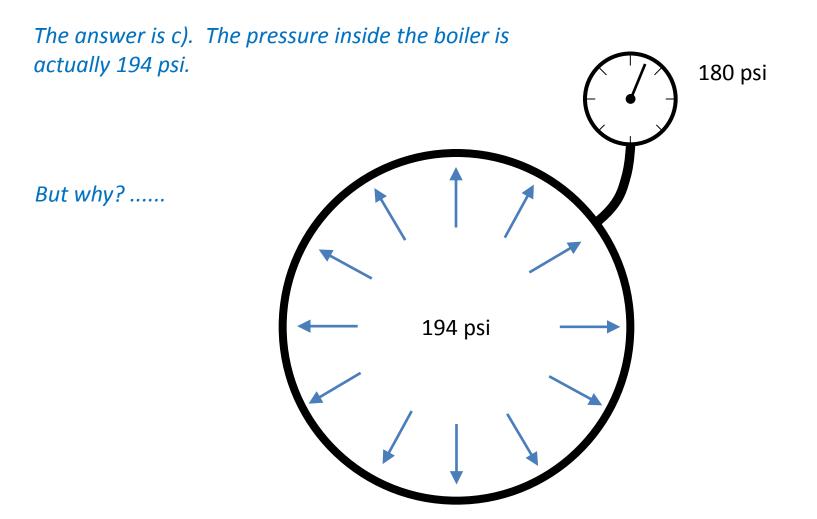
This will be explained further...

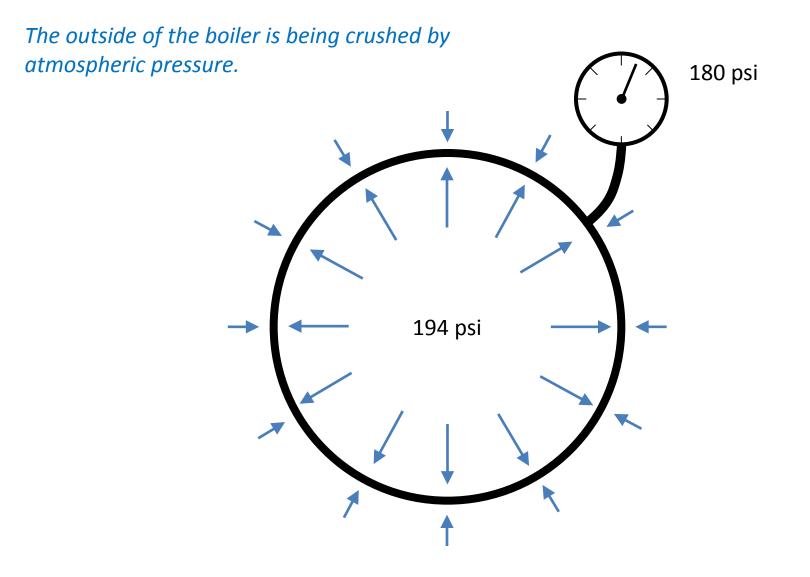
Consider a boiler and its pressure gauge.







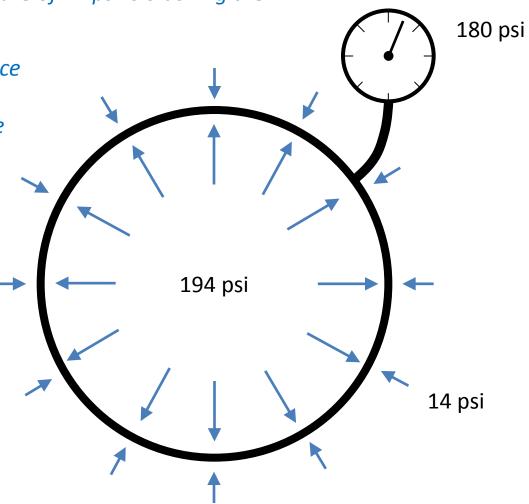




An atmospheric pressure of 14 psi is crushing the outside of the boiler.

180 psi is the difference in pressure between the outside and inside of the boiler.

There was already 14 psi inside the boiler before it was pressurised (when the gauge read zero).



Some useful notes:

180 psi is known as the gauge pressure, and in representing the pressure difference it correctly indicates the resultant pressure acting to try and burst the boiler.

194 psi is known as the absolute pressure.

Atmospheric pressure is typically 14.5 psi at sea level. For these notes it is taken as the round figure of 14 psi.

Let us now examine how this relates to vacuum brakes. There is a complication. Vacuum is not measured in psi. Vacuum is measured in inches...

A complete vacuum is when the air pressure is zero.

<u>Pressure</u>

Atmosphere

14 psi

Complete vacuum

0 psi

A complete vacuum is measured as 30 inches	<u>Pressure</u>	<u>Vacuum</u>
Atmosphere	14 psi	0 in
Complete vacuum	0 psi	30 in

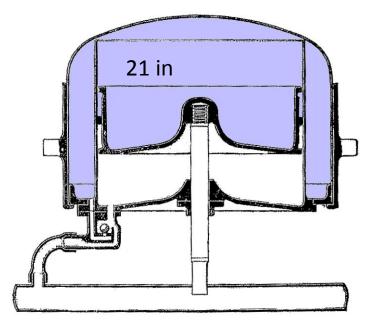
Note that vacuum is not negative pressure. The measurement of vacuum represents the reduction of air pressure.

Locomotive ejectors do not achieve a complete vacuum. Those of the Ffestiniog Vacuum Pressure *Railway generate 21 inches of vacuum.* 0 in Atmosphere 14 psi 30 in Complete vacuum 0 psi Brake vacuum 2 21 in

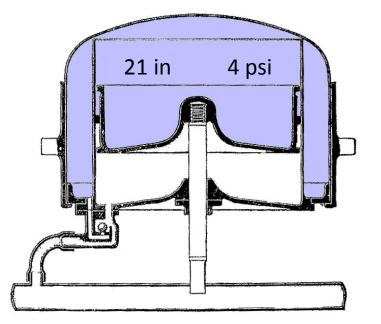
<u>Question:</u> what is the air pressure at 21 inches of vacuum?

Answer	<u>Pressure</u>	<u>Vacuum</u>
Atmosphere	14 psi	0 in
Complete vacuum	0 psi	30 in
Brake vacuum	4 psi	21 in

This is the vacuum brake cylinder shown with 21 inches of vacuum above the piston.

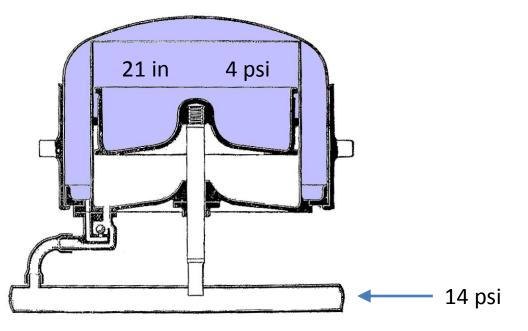


As shown previously, 21 inches of vacuum is actually 4 psi of air pressure.



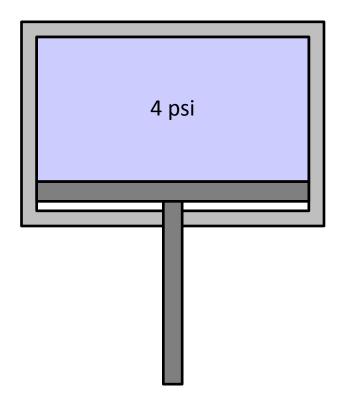
If train pipe vacuum is completely destroyed, atmospheric air at 14 psi is allowed into the brake pipe. This creates a pressure difference above and below the piston of 10 psi.

This 10 psi pressure difference, acting over the area of the piston provides the maximum brake force.



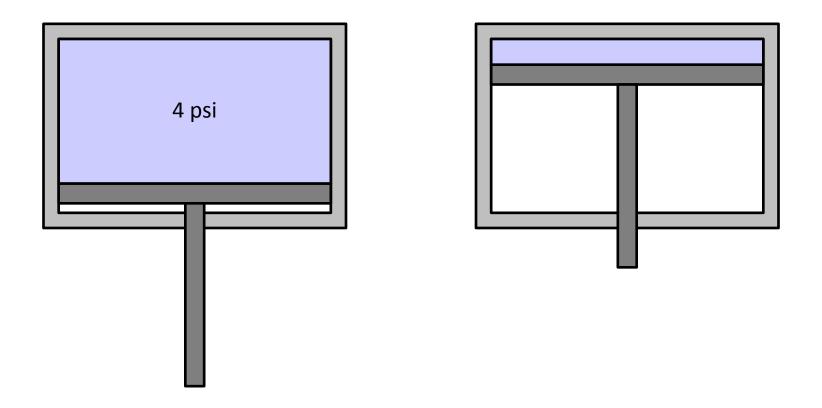
For reference, the pressure difference in air brake cylinders is around 50 psi. Therefore, the air brake system can use smaller cylinders to achieve the same brake force.

Now we will look at an important operational aspect of the vacuum brake cylinder. The diagram below represents a vacuum brake cylinder when the brake is released.

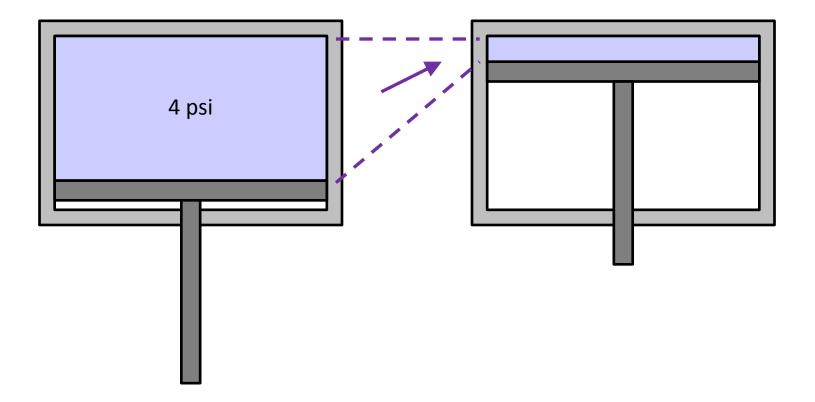


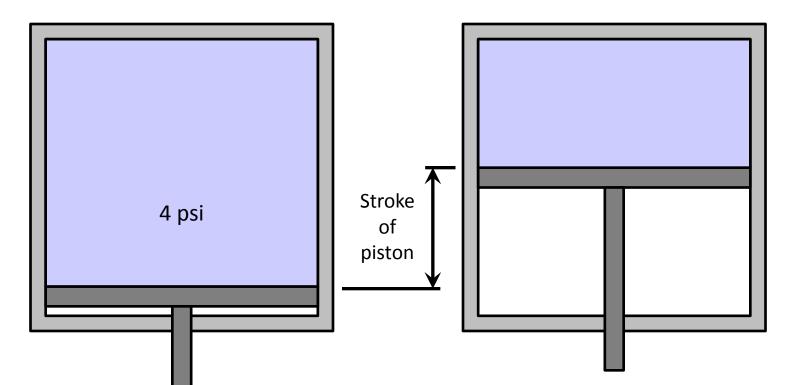
The diagram on the right represents the same cylinder when the brake is applied.

<u>Question:</u> Can you see the problem with this?

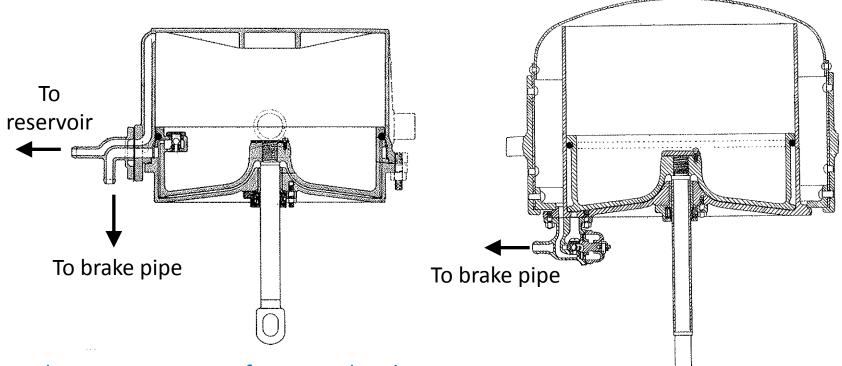


<u>Answer:</u> The air at 4 psi is being compressed into a smaller space as the piston moves up the cylinder. As a result, its pressure will increase. The pressure difference across the piston will consequently reduce, which in turn will reduce the brake force.





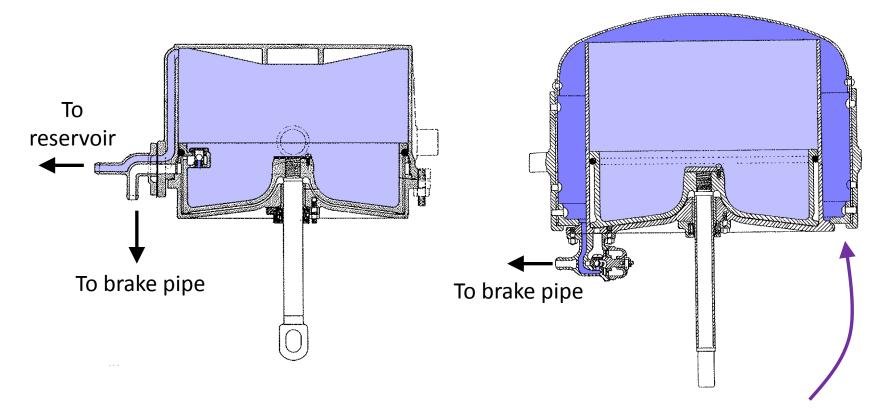
The problem cannot be completely avoided, but the effect can be minimised by increasing the volume of the space above the piston. Therefore, the compression of the air will be much less significant. The area above the piston is referred to as the **vacuum chamber**.



There are two types of vacuum chamber.

The cylinder shown above left is connected to a separate reservoir to increase the size of its vacuum chamber.

The cylinder shown above right contains its own annular reservoir.



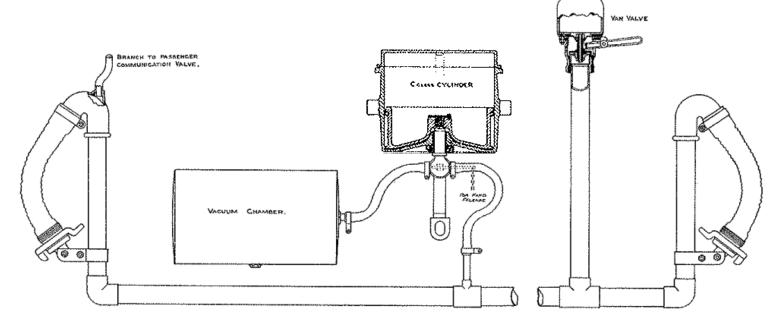
Although the vacuum chamber on the cylinder shown above right appears small, it is actually quite a large volume because it surrounds the outer edge of the working cylinder.

The system

SINGLE GAUGE

Vacuum brakes

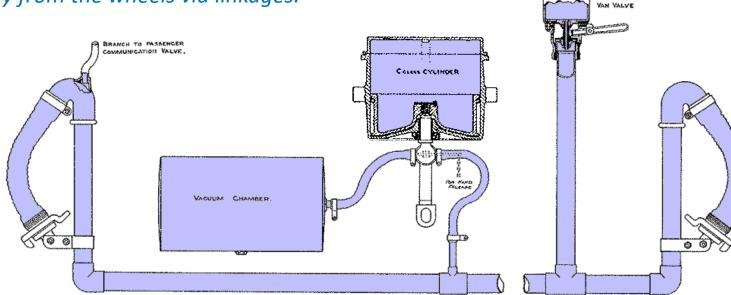
This diagram shows the complete system as typically fitted on trailing vehicles (coaches and wagons). The van valve is not always present. In its place may be the passenger operated emergency valve.



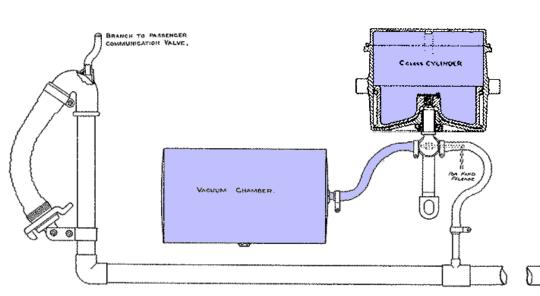
SINGLE GAUGE

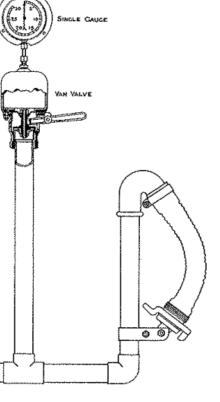
Vacuum brakes

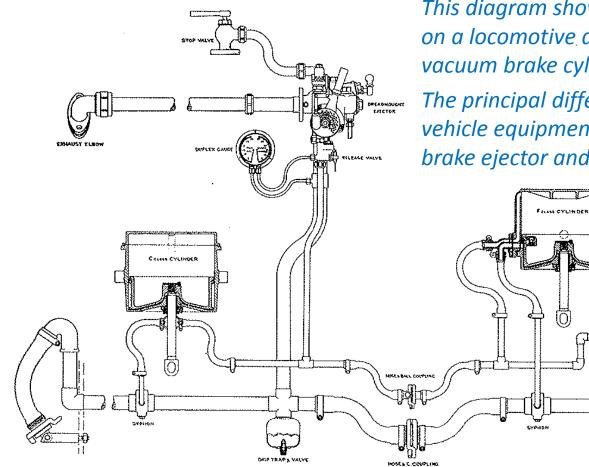
When the brake system is activated, and is in its release mode, a partial vacuum is generated in the train brake pipe and the vacuum chamber. The brake piston falls under gravity and moves the brake blocks away from the wheels via linkages.



When the brake is applied, atmospheric air pressure fills the brake pipe but cannot enter the vacuum chamber due to a non-return valve. A pressure difference is generated across the brake piston, which will pull the brake blocks onto the wheels via linkages.







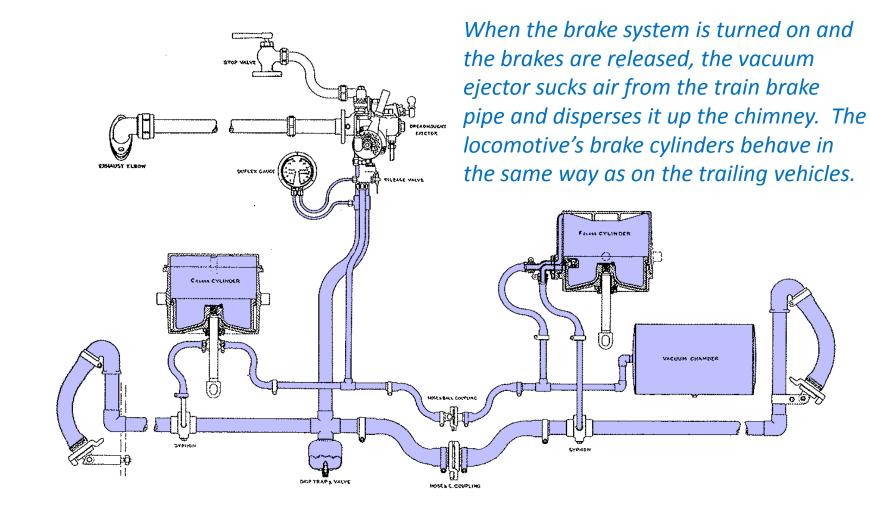
This diagram shows a typical system as on a locomotive and tender fitted with vacuum brake cylinders.

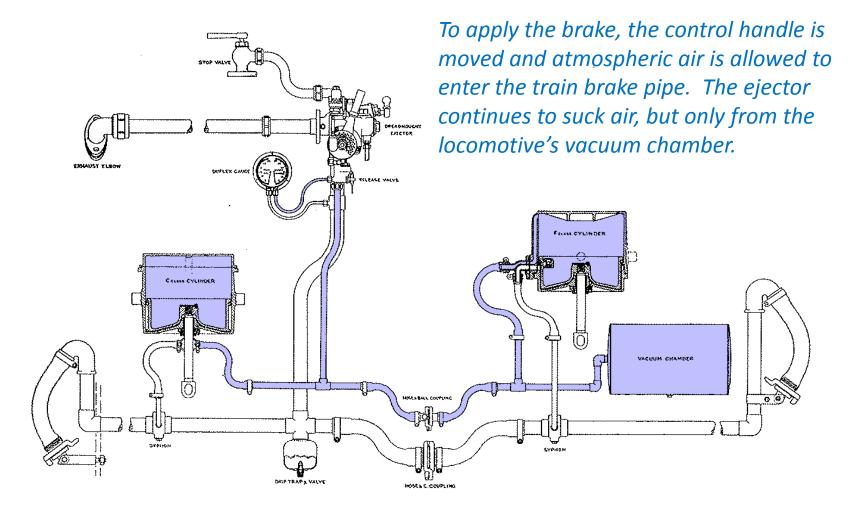
The principal difference from the trailing vehicle equipment is the presence of the brake ejector and control valve.

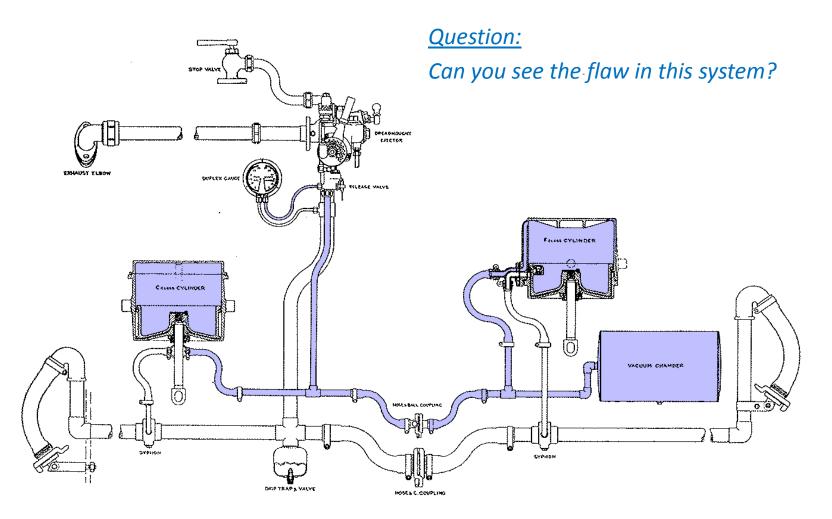
VACUUM CRAMBER

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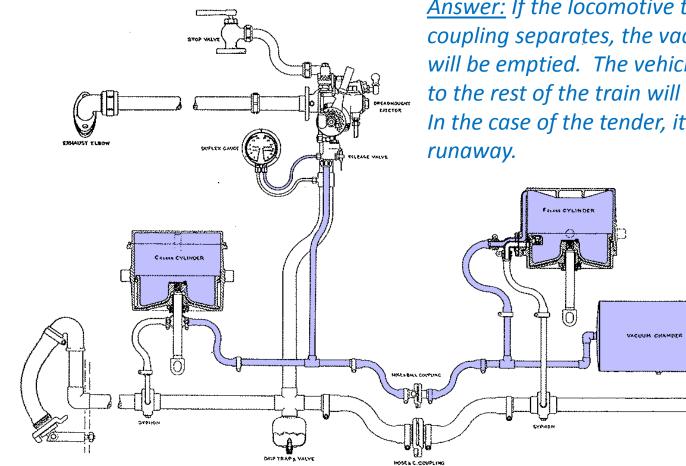






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Vacuum brakes

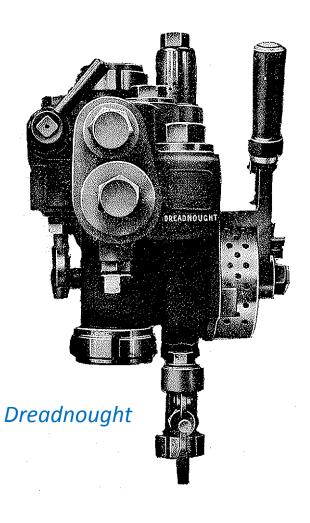


Answer: If the locomotive to tender coupling separates, the vacuum chamber will be emptied. The vehicle not coupled to the rest of the train will have no brake. In the case of the tender, it will become a

In terms of continuous brakes, the locomotive and its tender are considered as one vehicle because they are permanently coupled.

Therefore, if you are working on a locomotive restoration, give due consideration to the integrity of the locomotive to tender coupling.

Brake control valves



These two types of brake controller are used extensively on the Ffestiniog Railway.

<u>*Question:*</u> What is the fundamental difference between them?

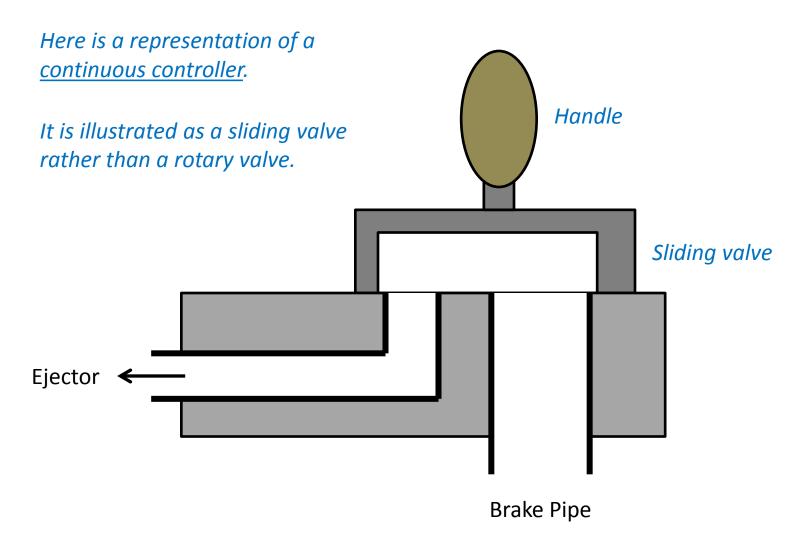


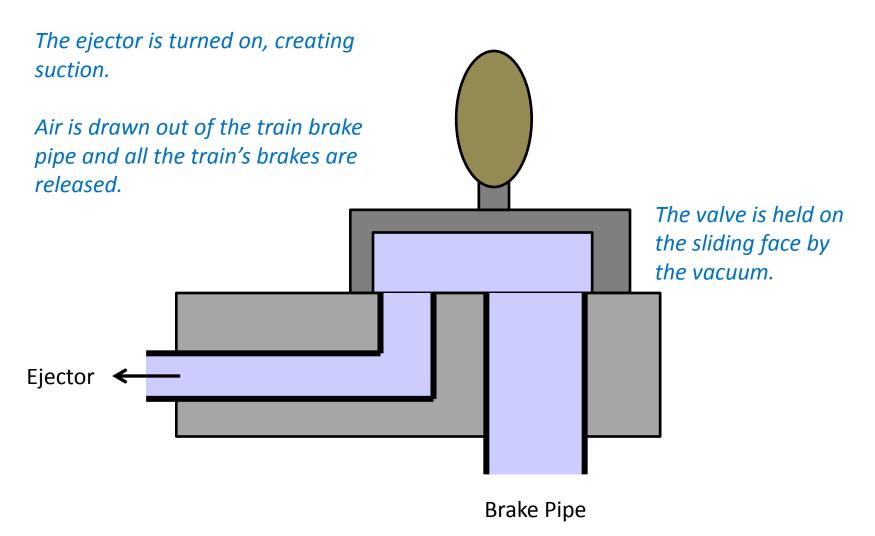
DMU valve

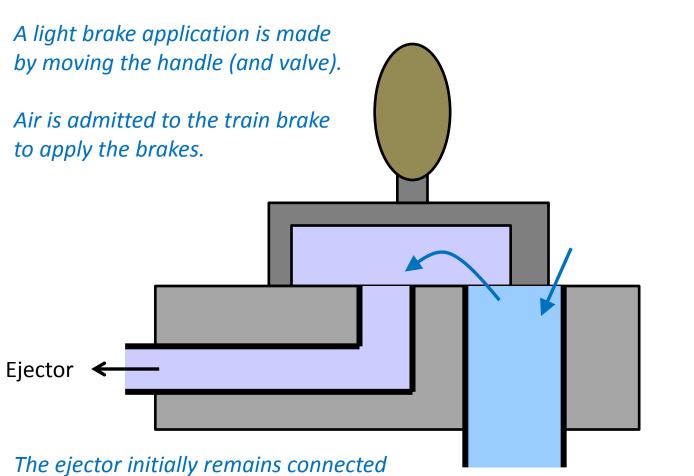
<u>Answer:</u> the Dreadnought has a continuous operation whereas the DMU valve has a lap position.

Another difference is that the Dreadnought contains the ejector in the same fitting as the controller, whilst the DMU valve requires a separate exhauster.

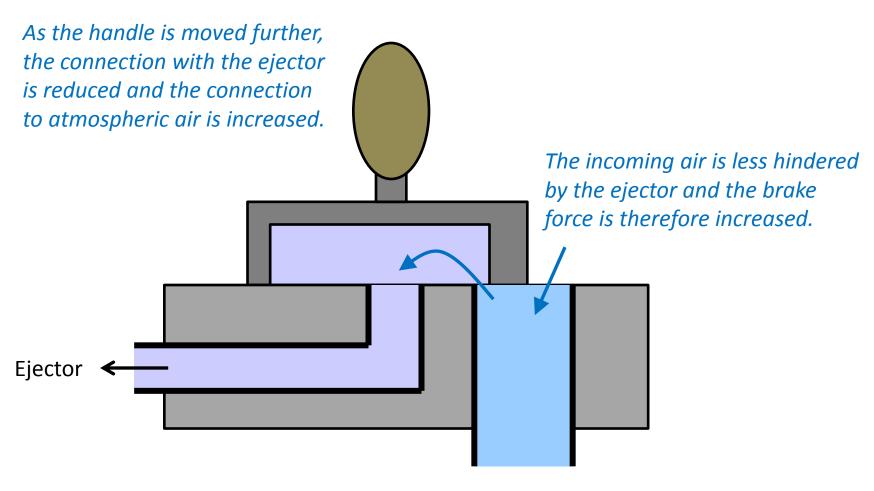
The next few slides will look at the difference between a continuous controller (like the Dreadnought) and a lap position controller (like the DMU valve).

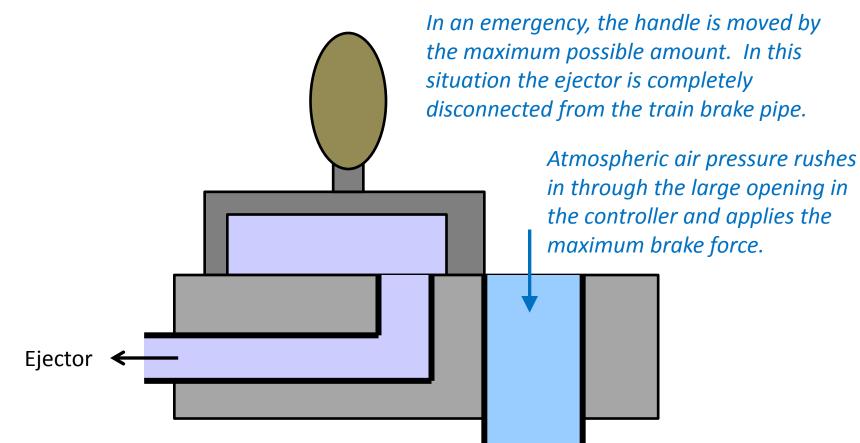




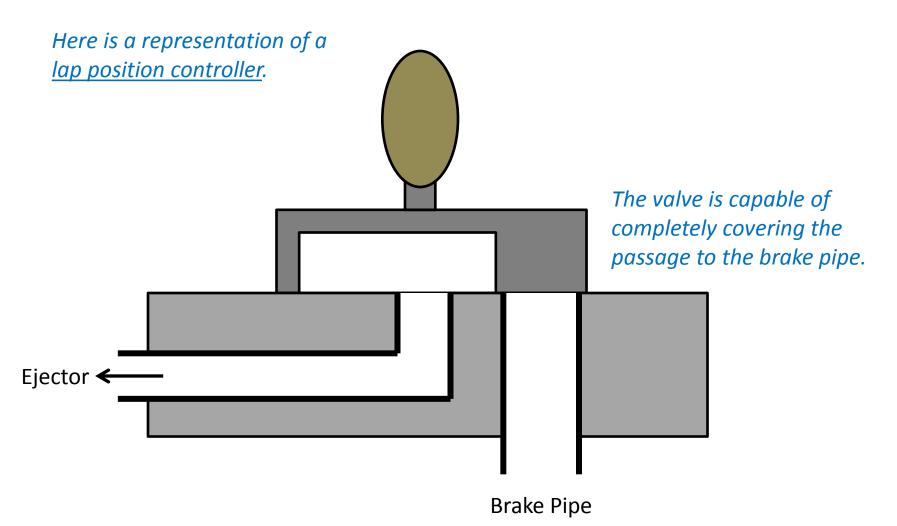


to the brake pipe and partially counters the incoming air.

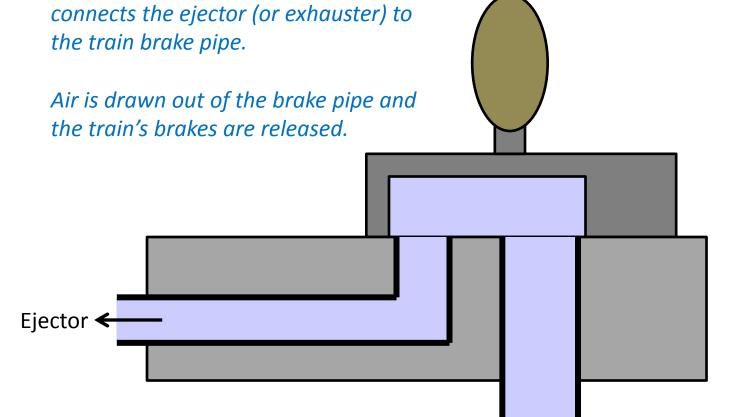


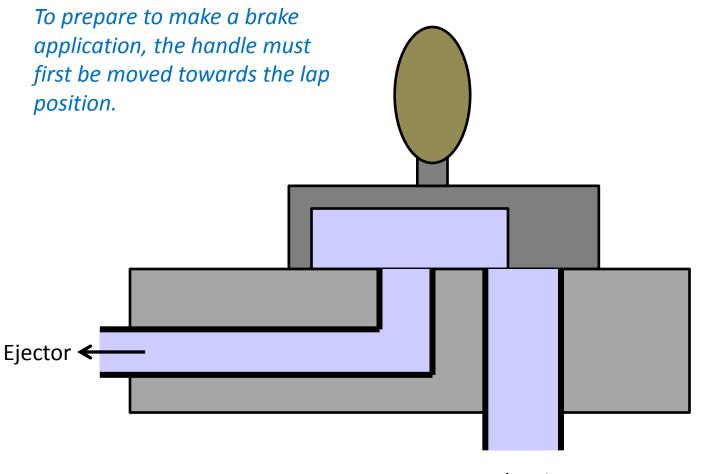


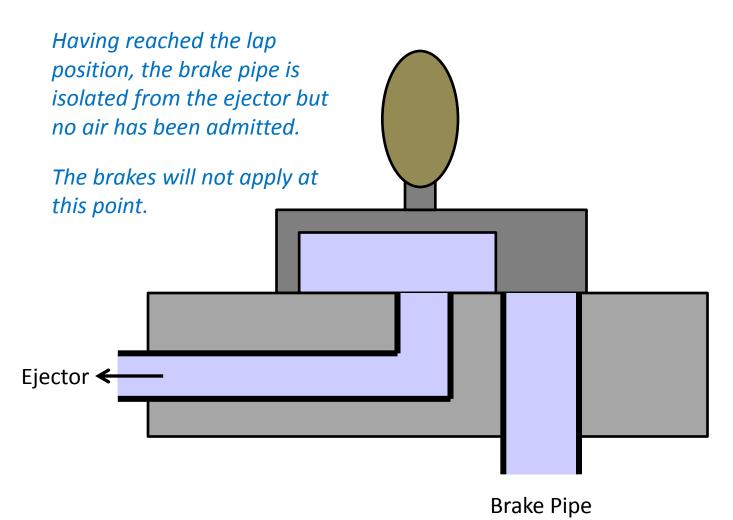
Operation of a continuous type of controller is intuitive. The further the brake handle is moved the greater the braking force will be.

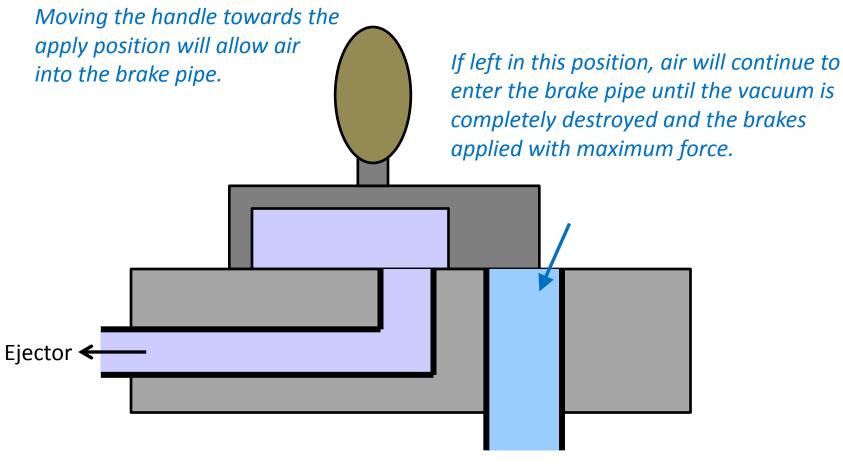


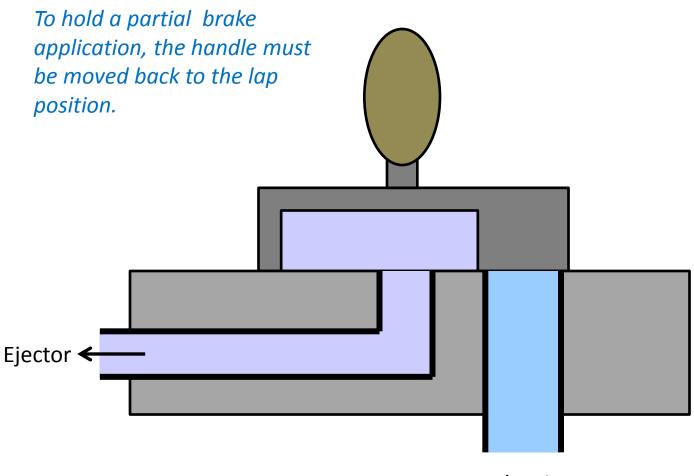
In the release position the brake valve the train brake pipe.

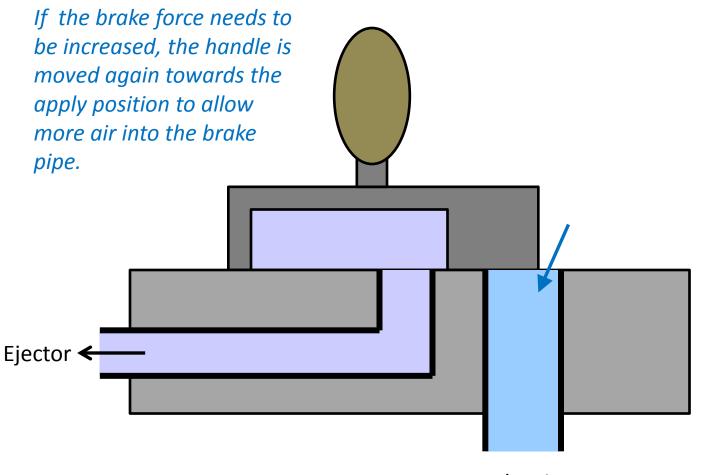


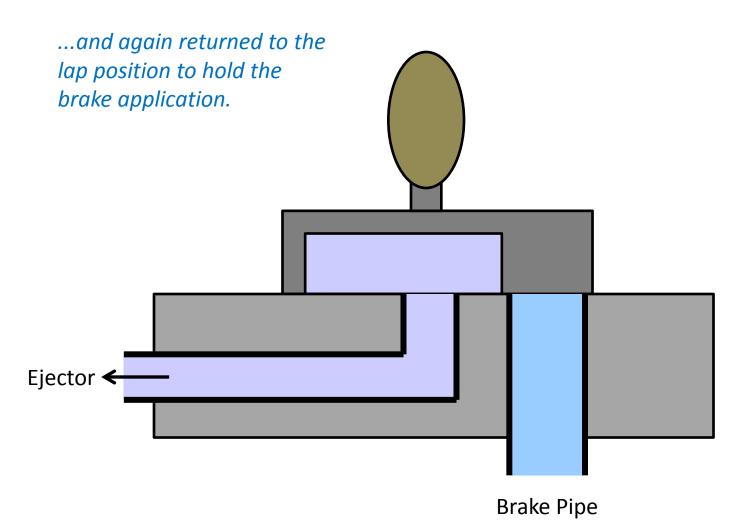


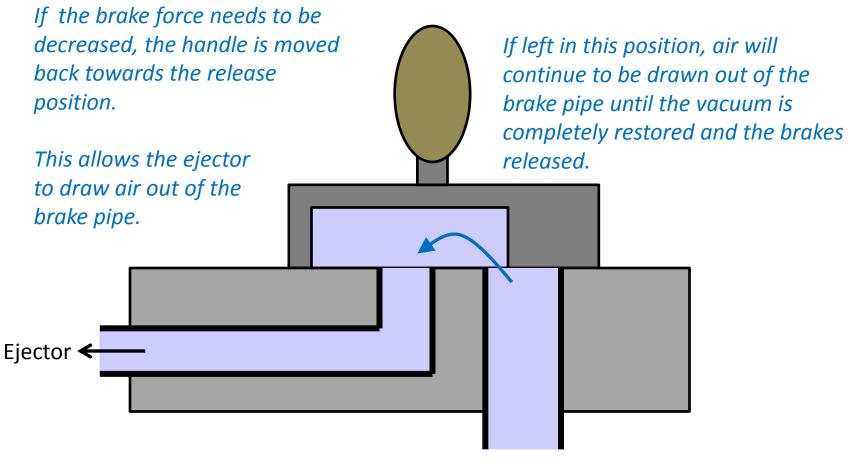


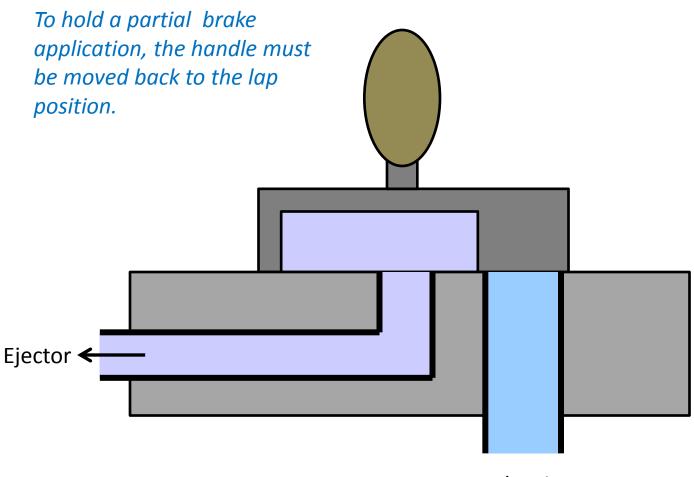


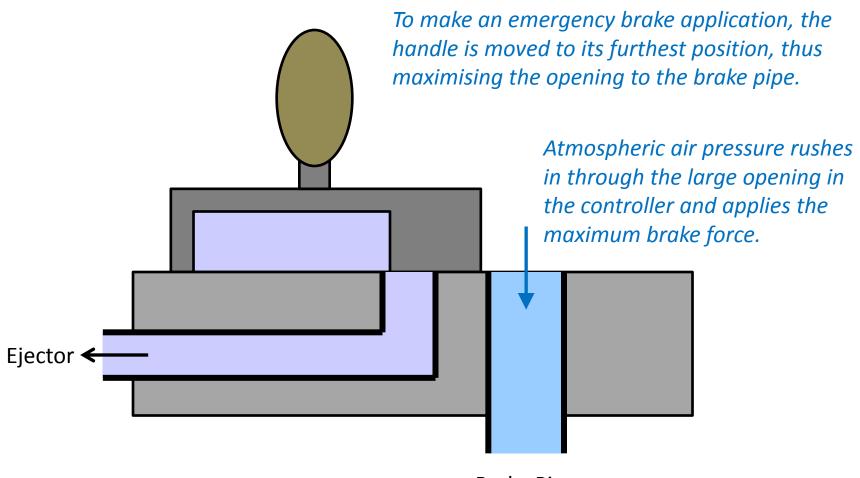












Brake Pipe

Important note:

Whilst the brake controller is in the lap position, a small amount of air will enter the brake pipe as leakage at all the joints in the system. These leaks cannot be countered by the ejector, which is now disconnected from the brake pipe. Therefore, a gradual increase in brake force may be experienced whilst in the lap position.

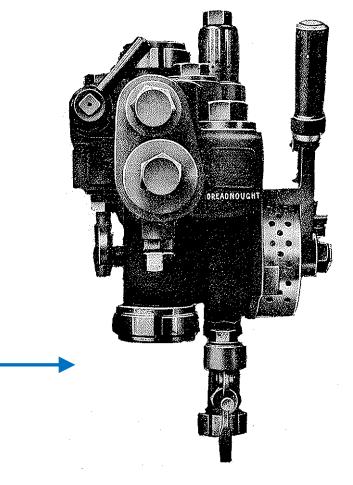
The controller <u>must not</u> be left in the lap position when the train brakes have been released. This is because air could leak into the vacuum chambers of the trailing vehicles, resulting in reduced brake effectiveness. The ejector <u>must</u> remain connected to the train brake pipe up until a brake application is required, so that the vacuum chambers are all fully charged.

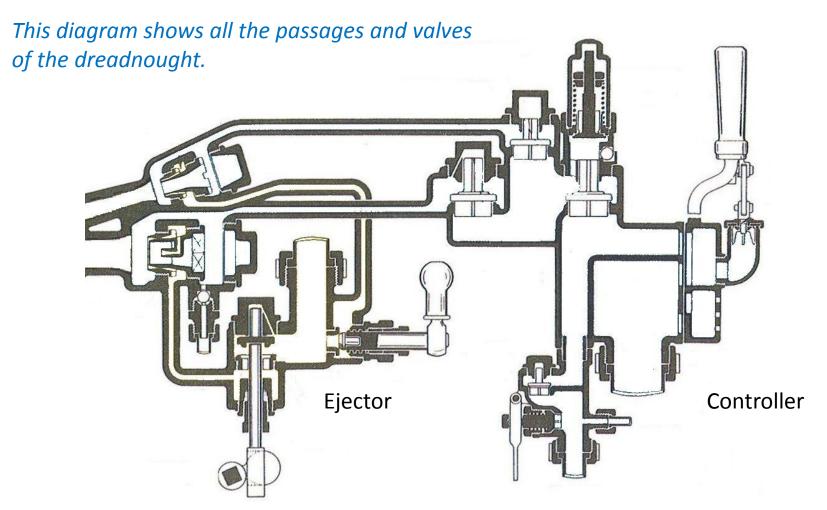
The next section of slides will examine the actual internal configuration of the vacuum brake controllers and ejectors used on the Ffestiniog and Welsh Highland Railways.

The dreadnought is typical on the Ffestiniog Railway steam locomotives.

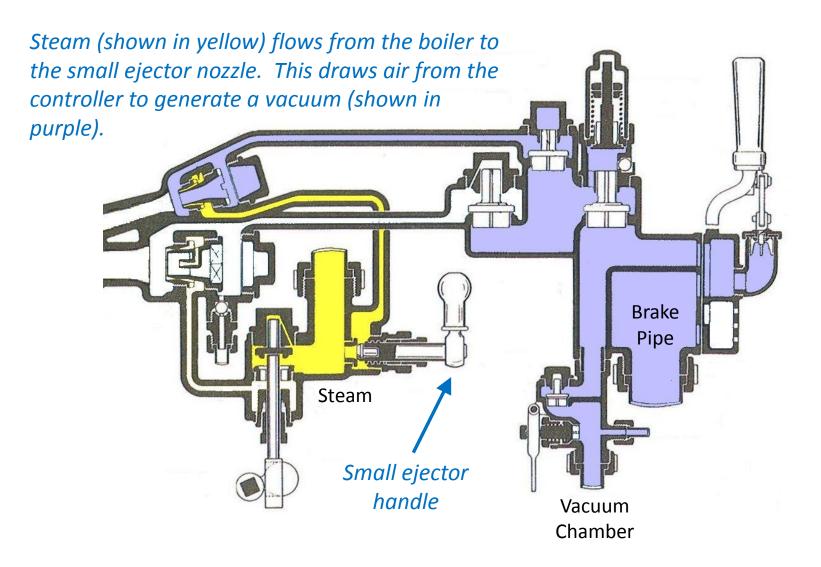
The SJ brake controller is typical on the Welsh Highland Railway locomotives.

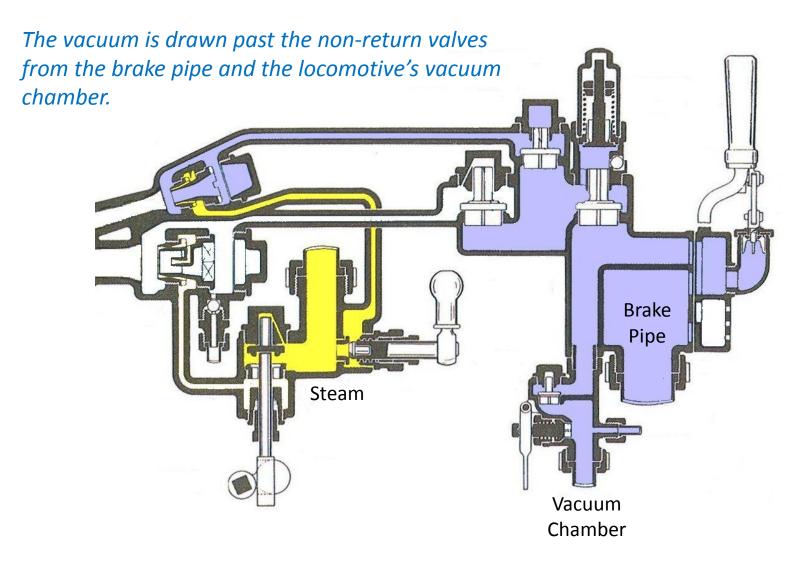
The <u>dreadnought</u> is examined first, being essentially as described by the previous description of a continuous controller.

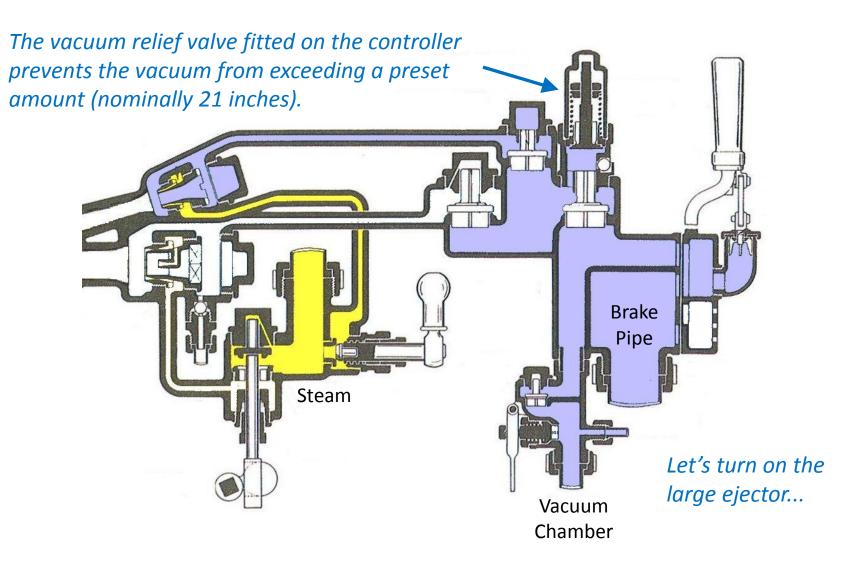


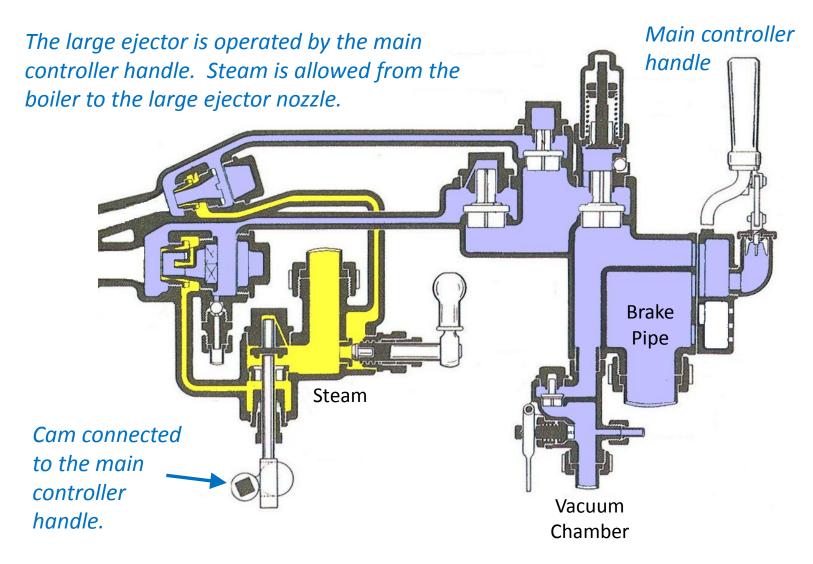


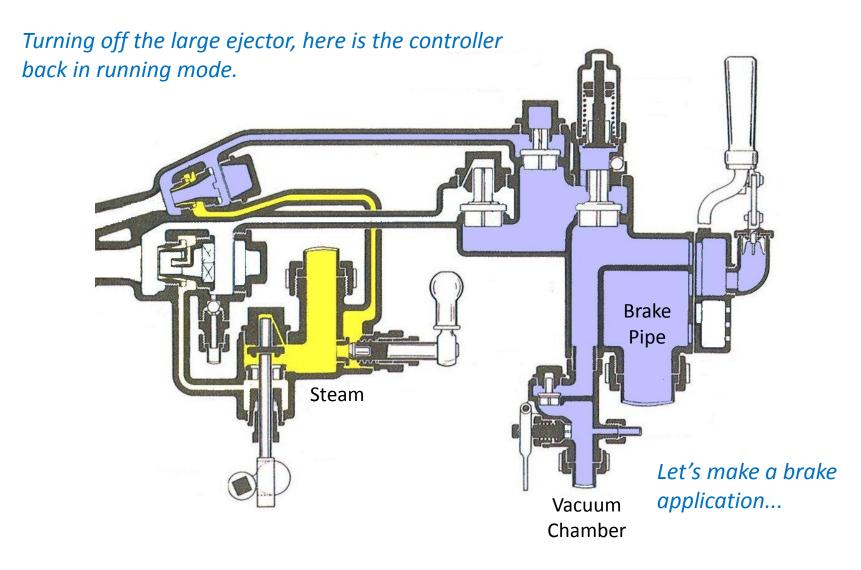
Let's turn on the small ejector...

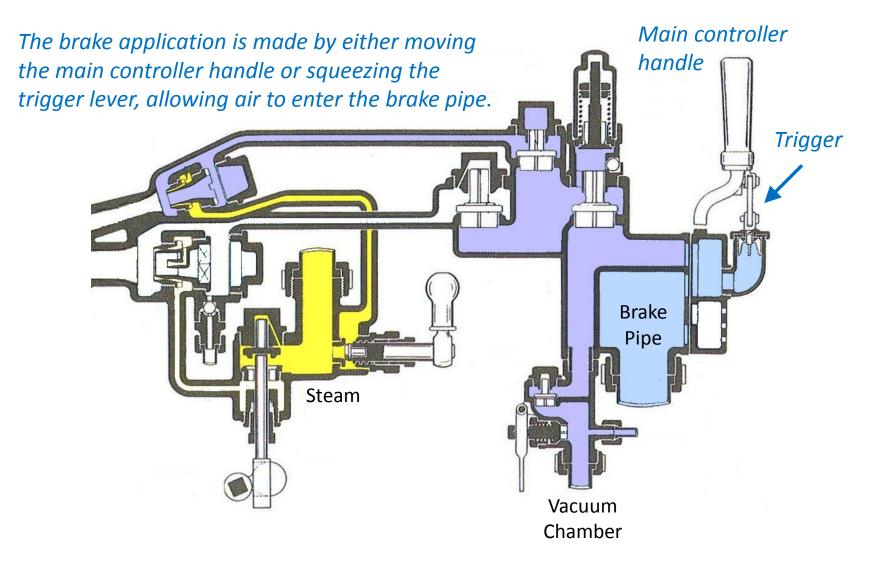


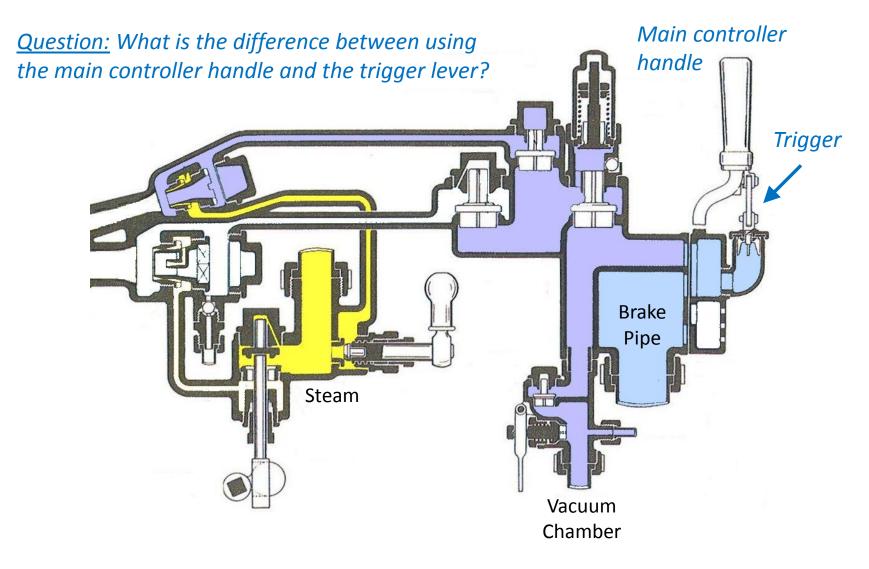










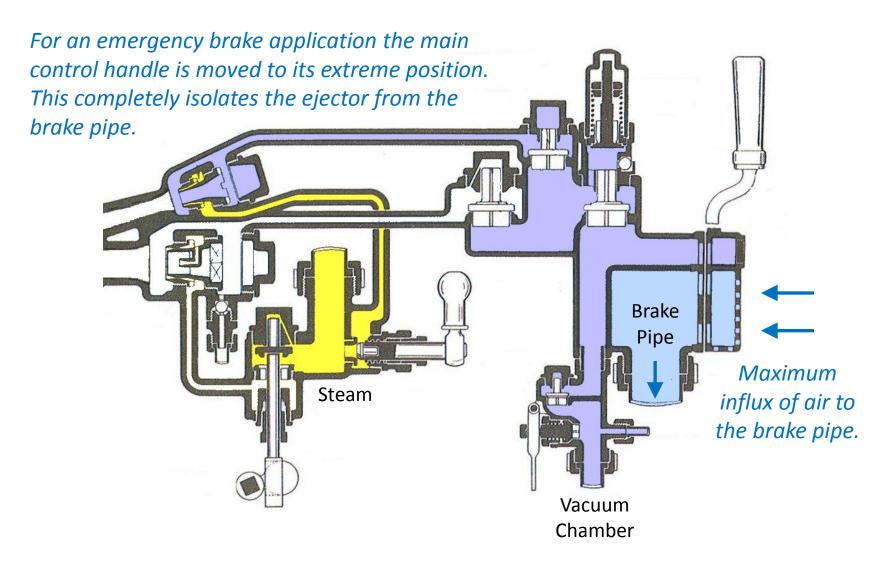


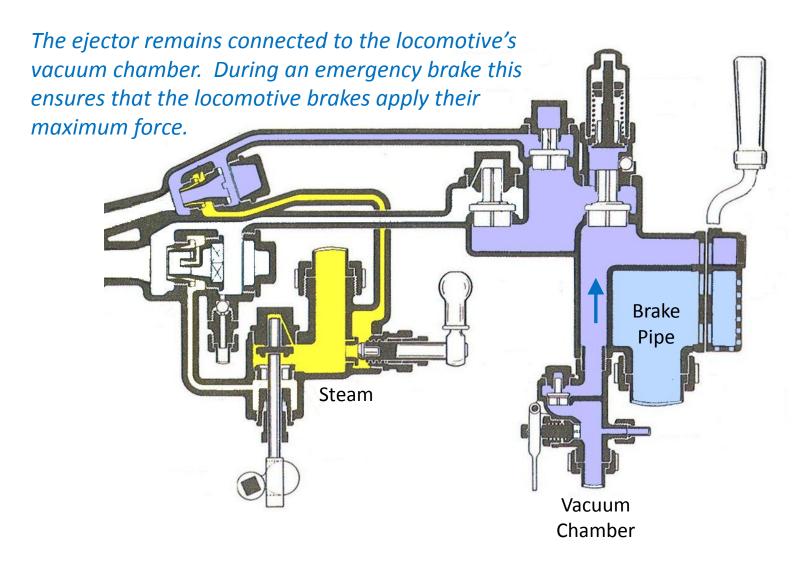
<u>Answer:</u>

Moving the main controller handle will start to cut off the connection between the ejector and the brake pipe.

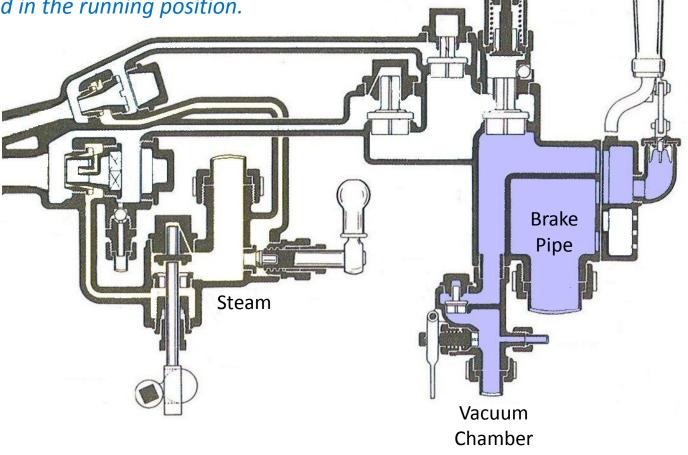
Squeezing the trigger allows only a small amount of air to enter the controller through the poppet valve, whilst the ejector will continue to draw air from the brake pipe.

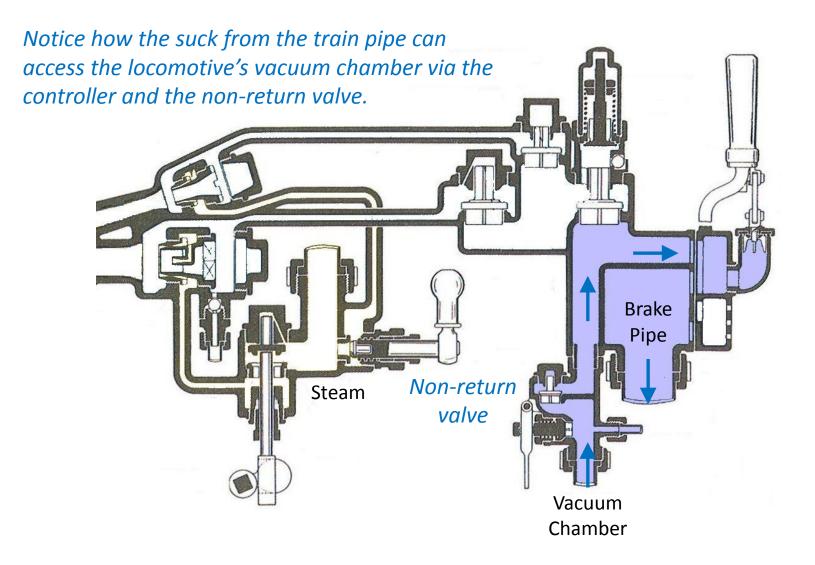
Hence, the trigger provides finer control of the air entering the brake pipe during light applications. However, it is unable to let enough air into the brake pipe for a harsh brake application.



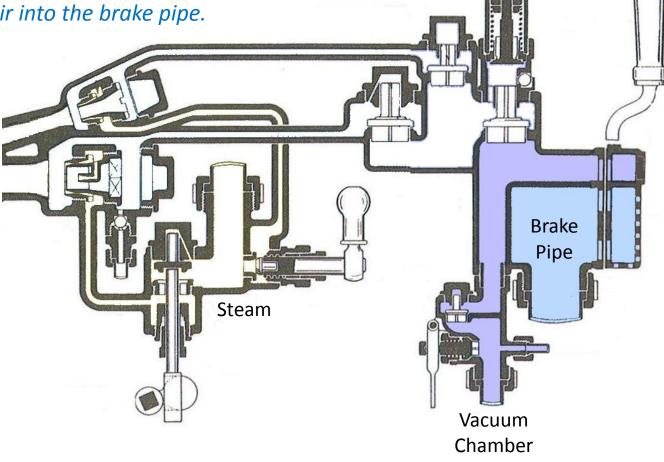


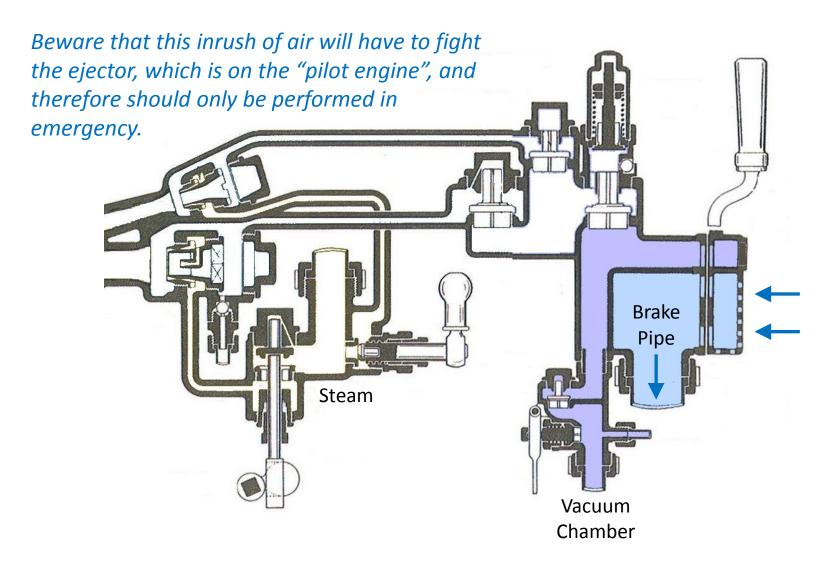
When double heading, the ejector of the "train engine" is turned off and the main control handle is placed in the running position.



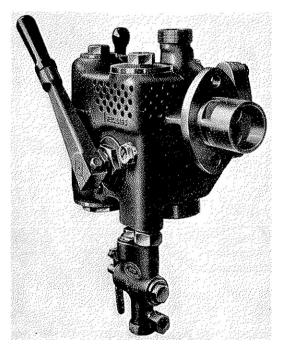


A brake application can still be made by the "train engine", by moving the main controller to allow air into the brake pipe.





Now we will examine the <u>SJ controller</u>. The version on the left is fitted on K1; the version on the right is fitted on the NGG16s.



<u>Question</u>: what are the differences?



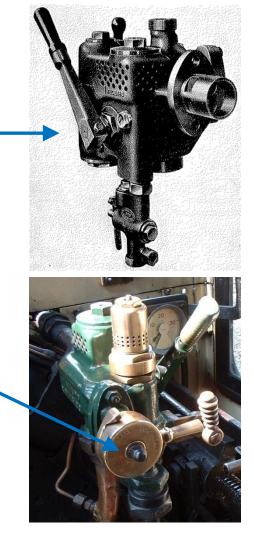
<u>Answer</u>:

The standard type of SJ controller as fitted on K1 incorporates the ejector as part of the same fitting.

The modified version on the NGG16s comprises only the controller mechanism, there being a separate ejector housing on the side of the boiler.

Further, the NGG16 controller incorporates a locomotive parking brake valve, which will be discussed later.

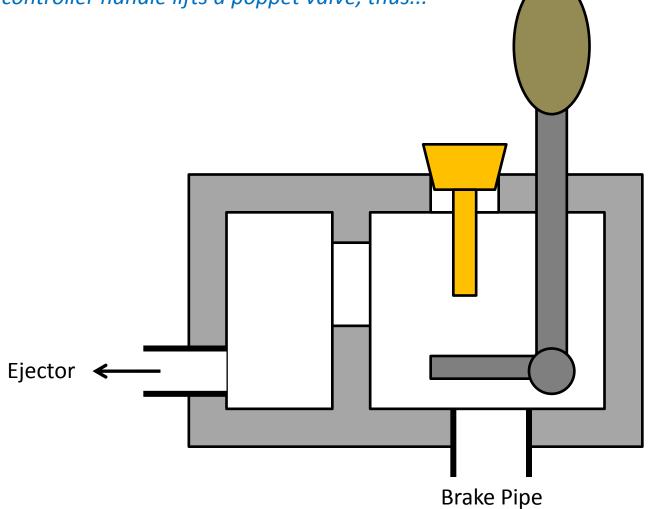
The essential controller mechanism is the same in both of these controllers.

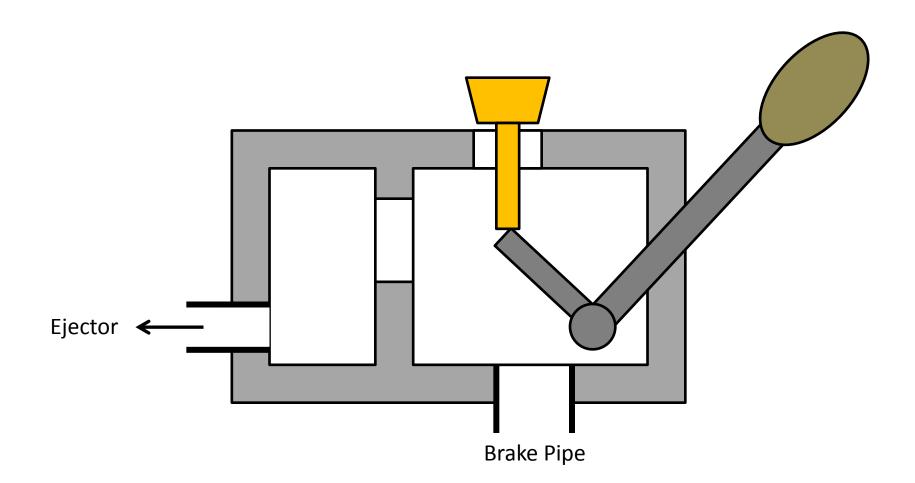


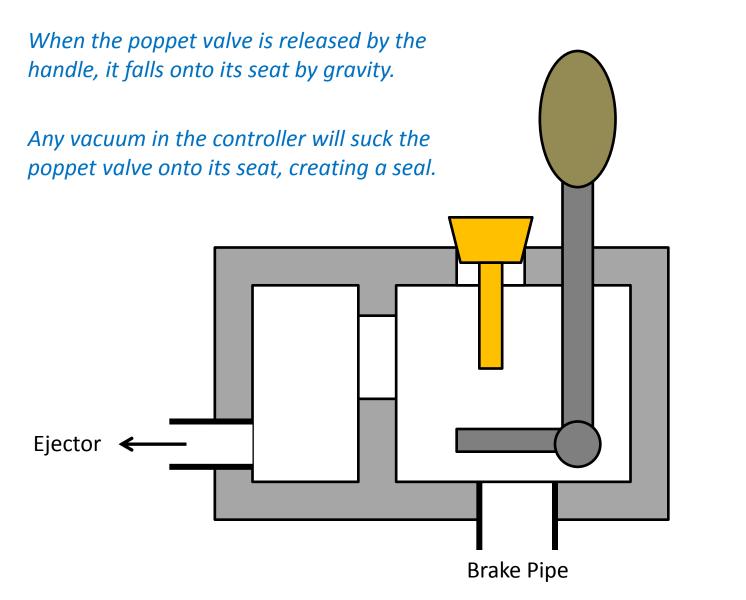
The SJ controller, shown here, is considerably more complex than the dreadnought.

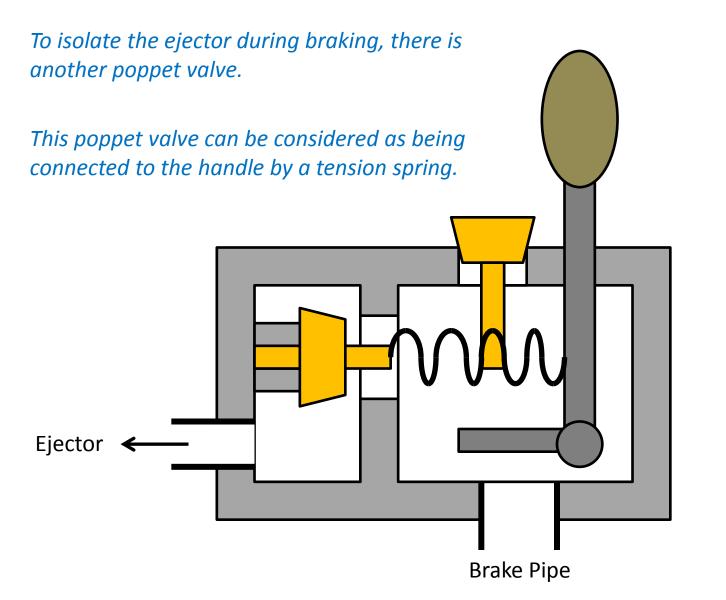
Therefore, we will start with a simpler diagram...

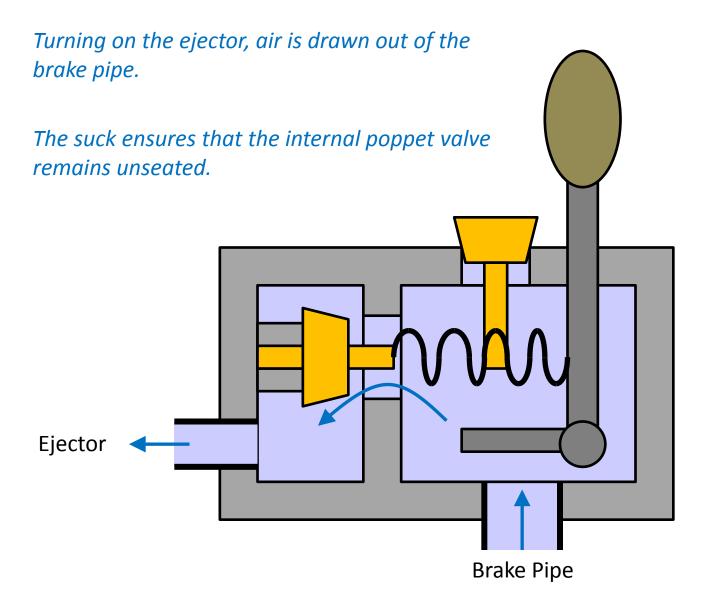
To allow air into the brake pipe, the main controller handle lifts a poppet valve, thus...



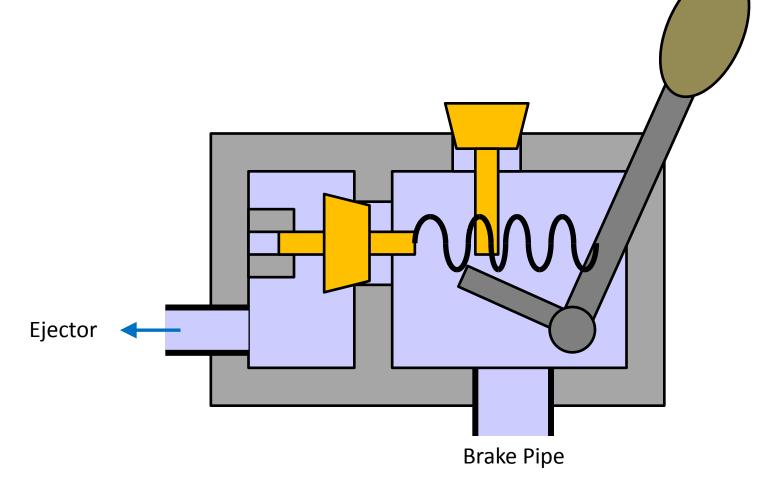




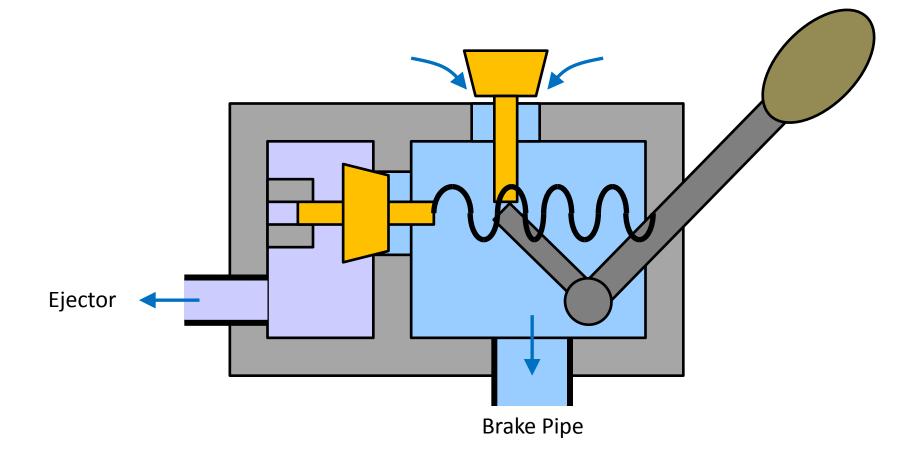




When a brake application is made by moving the main controller handle, tension is generated in the spring which pulls the internal poppet valve shut.

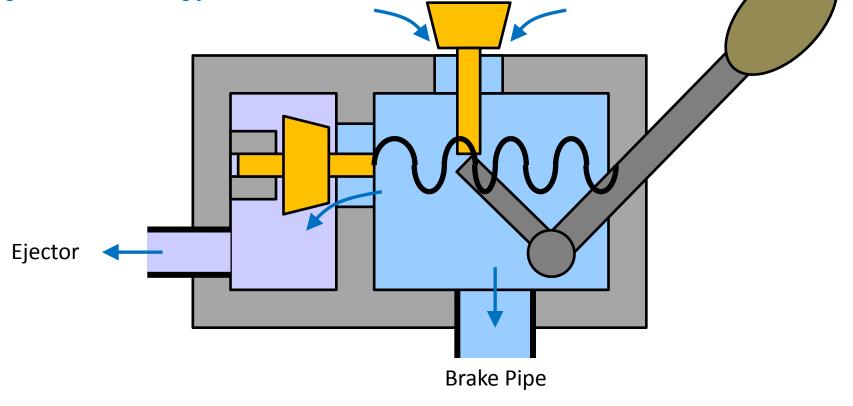


Further movement of the handle will force open the outer poppet valve, allowing air to enter the train brake pipe.



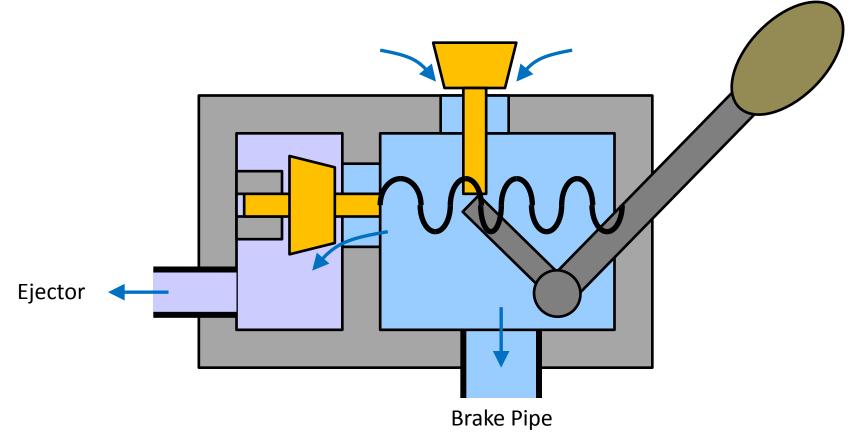
If the handle is held in this position, the air pressure will eventually overcome the spring and escape via the ejector.

The further the handle is moved, the greater the tension in the spring, the greater the required air pressure to overcome the spring, and the greater the braking force.



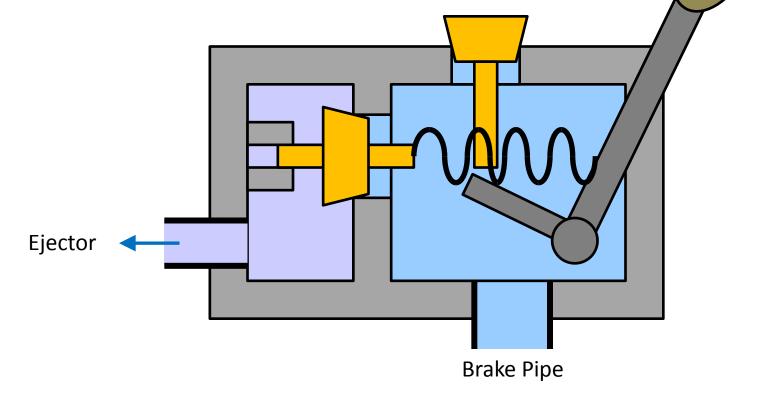
Hence, the SJ is a continuous type of controller. The further the handle is moved the greater the braking force will be.

However, it also has a lap position...

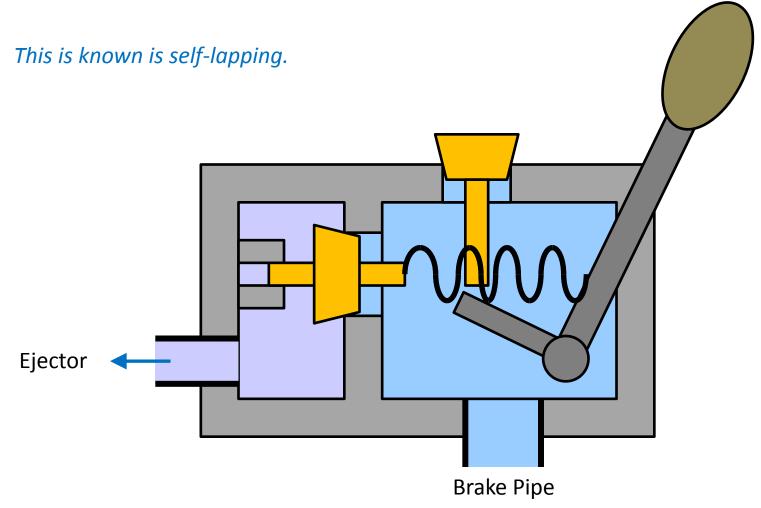


After initiating a brake application the handle can be moved back towards release so that it allows the outer poppet valve to close, but retains tension on the spring and internal poppet valve.

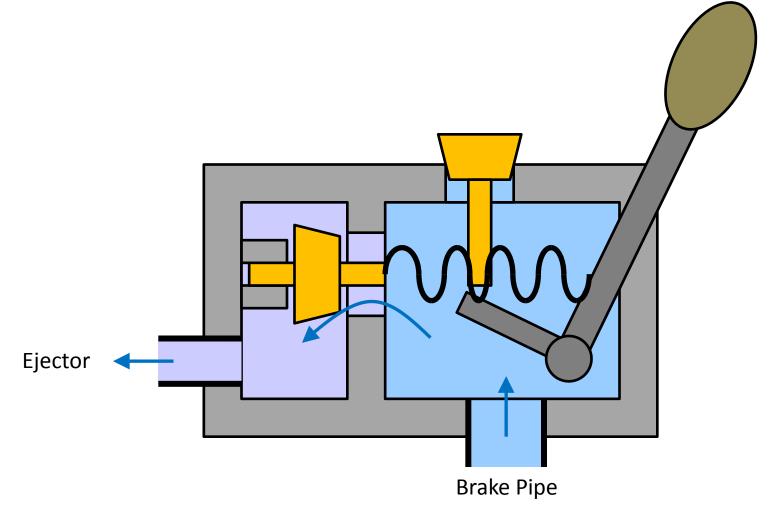
No more air can enter the brake pipe through the outer poppet valve, so the brake application is held constant.



Due to the internal spring and poppet valve, this lap position will actually compensate for any leakage in the brake pipe system.

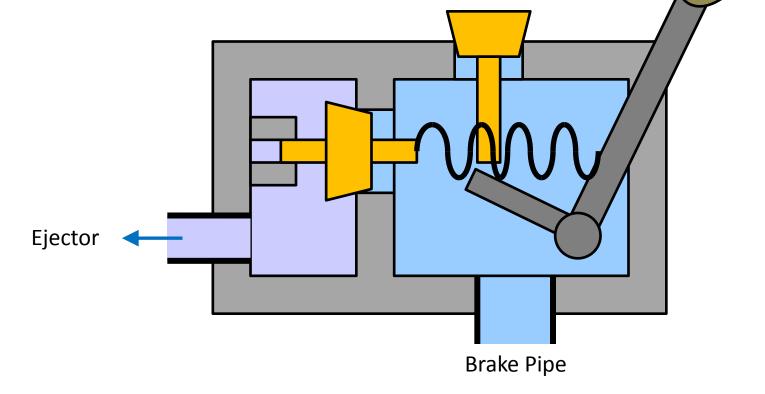


Any increase in brake pipe pressure will overcome the spring force, lift the internal poppet valve and escape via the ejector.



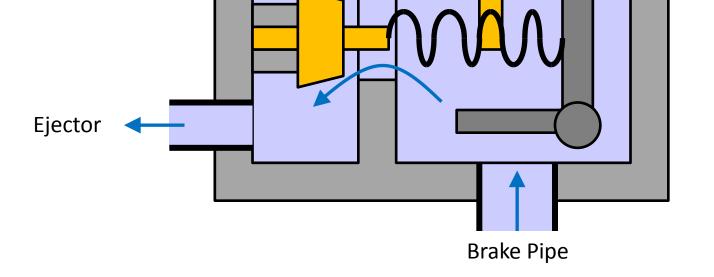
When the brake pipe pressure (or partial vacuum) is restored, the spring pulls the valve shut, isolating the ejector once again.

Note that this self-lapping position only works for light brake applications, because only a small amount of tension can be generated in the spring before lifting the outer poppet valve.



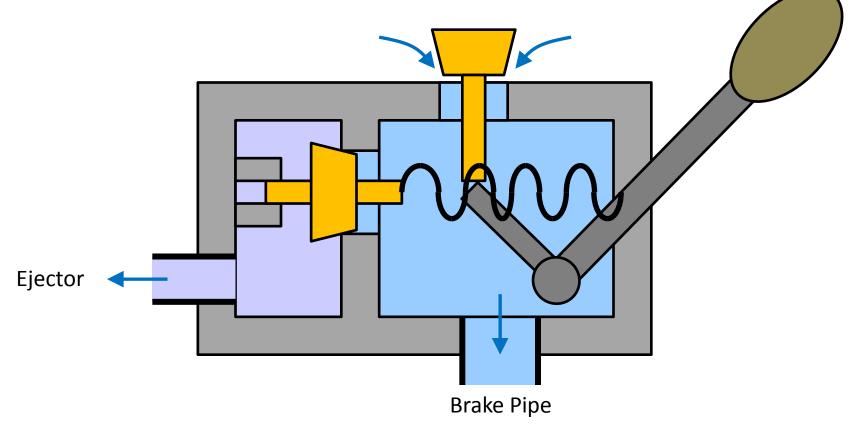
When it is required to release the brakes, the handle is allowed to return to its running position, and the spring tension is released.

The ejector is now able to generate the maximum brake pipe vacuum, drawing air through the internal poppet valve.



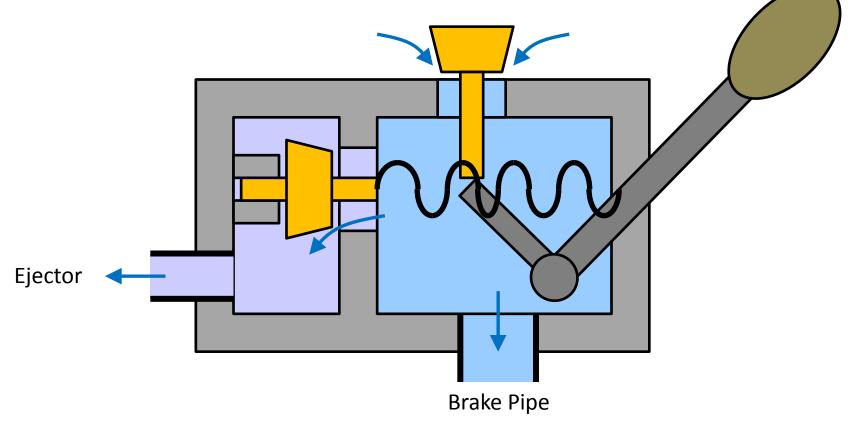
In an emergency, the handle is pulled fully back. This allows the maximum amount of air to enter the brake pipe through the outer poppet valve.

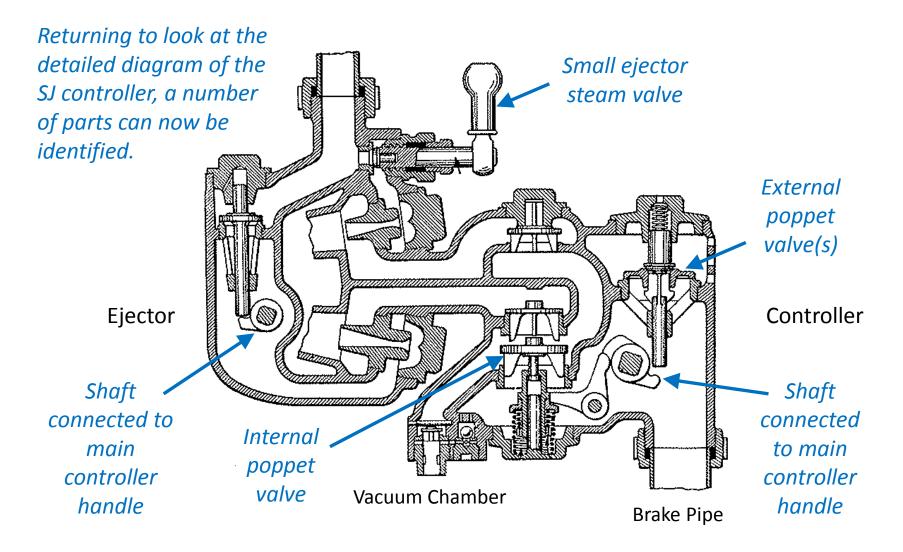
The handle generates maximum tension on the spring, holding the internal poppet valve shut and isolating the ejector from the brake pipe.

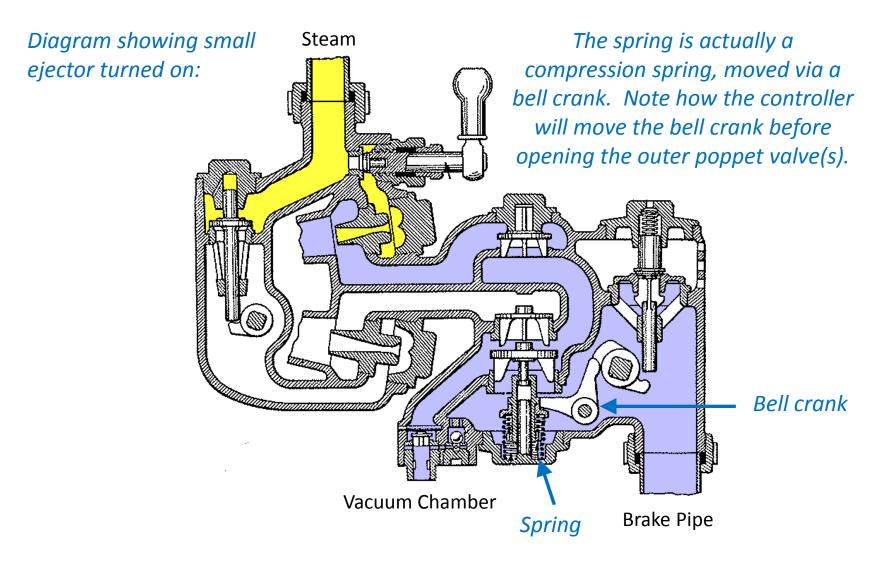


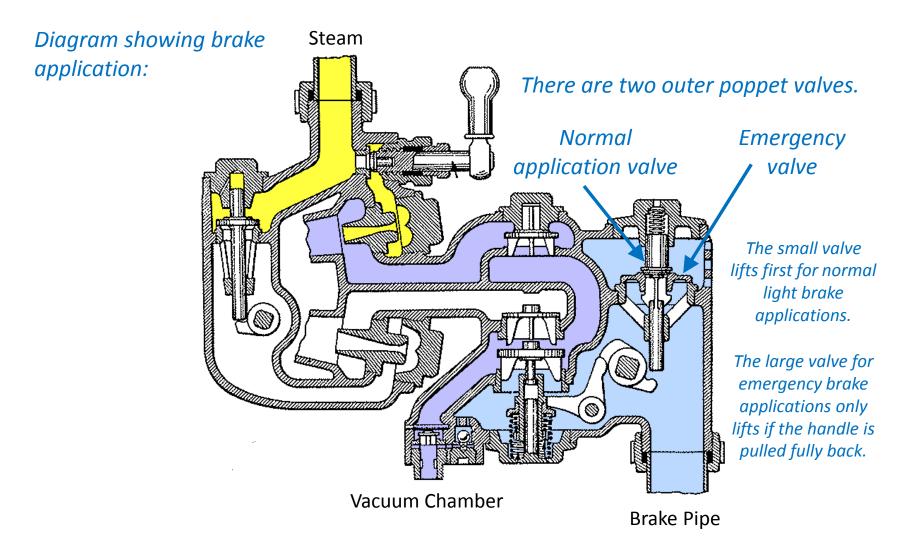
Once the brakes are fully applied (brake pipe vacuum fully destroyed), the incoming air will overcome the spring and force open the internal poppet valve.









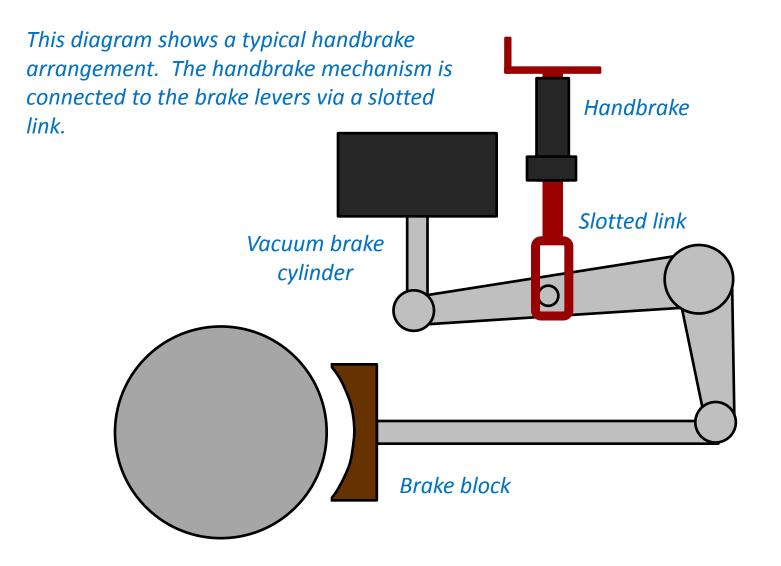


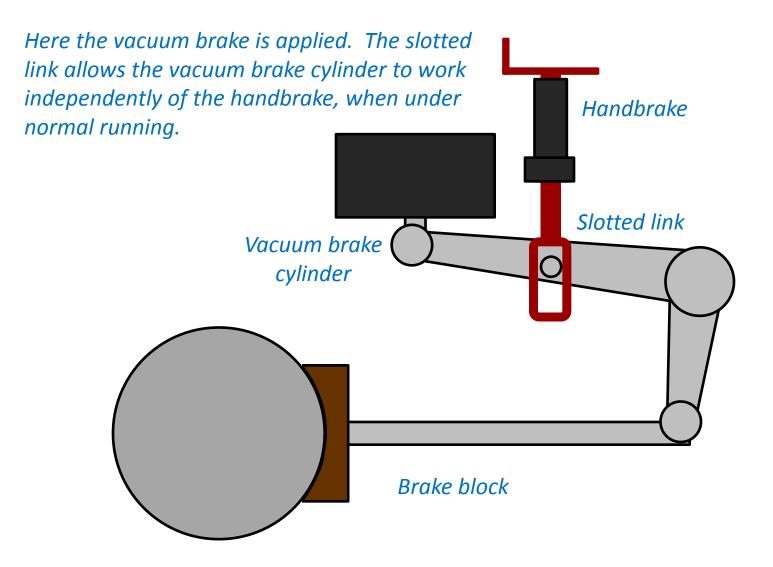
Independent brakes

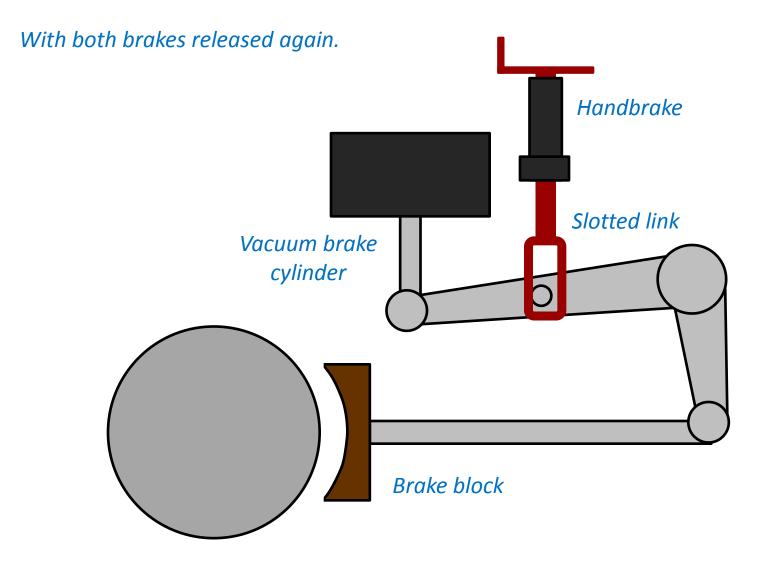
Independent brakes

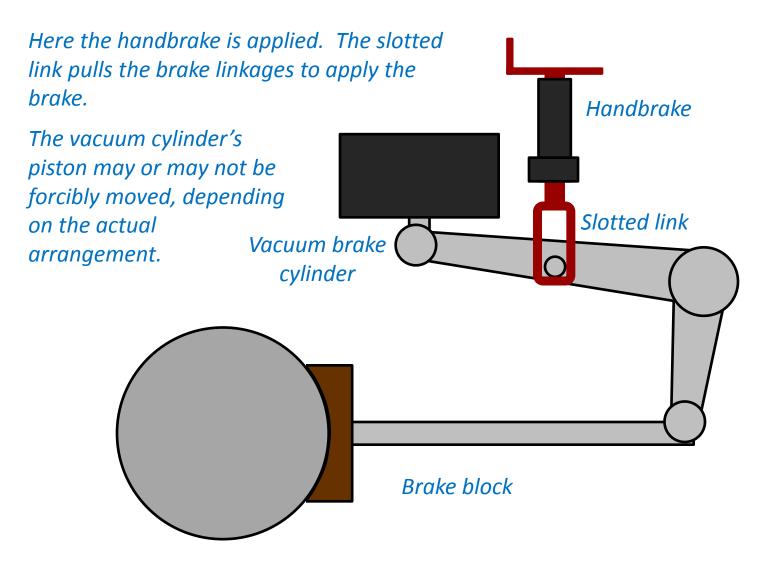
Besides the vacuum brake system, most locomotives have supplementary brakes that work either in conjunction with the vacuum brake system or are completely independent. The following systems will be examined in the following slides.

- Handbrake
- Vacuum parking brake (fitted on NGG16 locomotives)
- Steam brake
- Air brake (fitted on the Diesel locomotives)









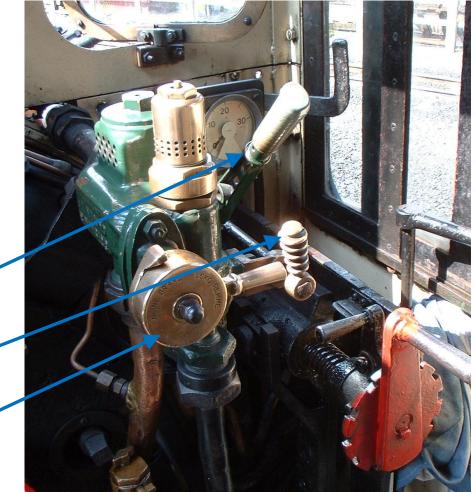
The next few slides explain the principle of the locomotive parking brake used on the NGG16s.

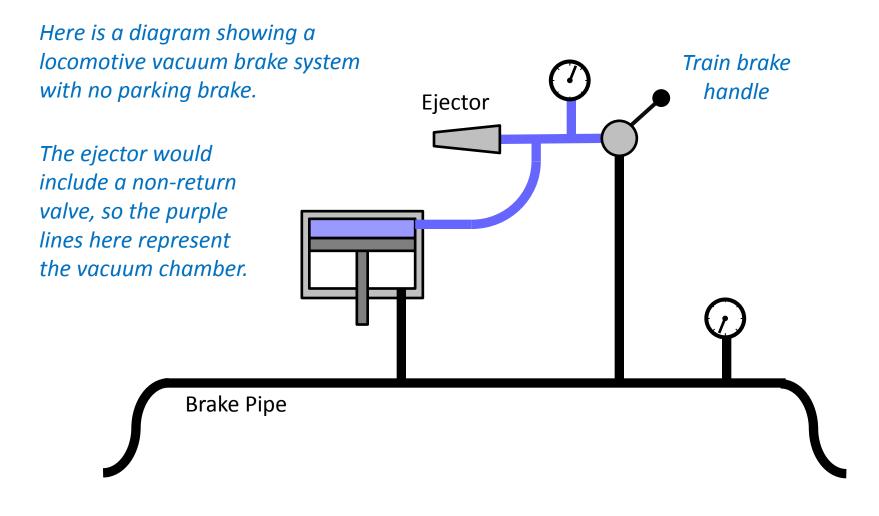
This is a vacuum operated parking brake system, which is unusual in the UK.

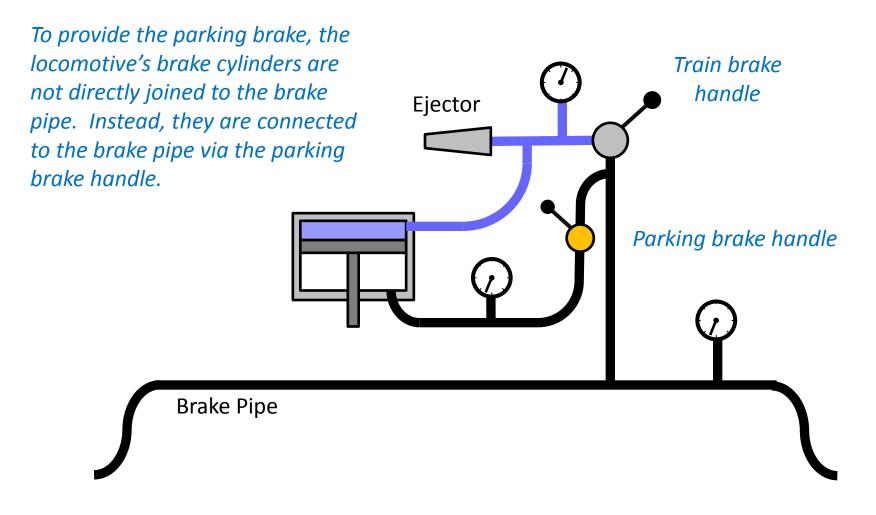
Train brake handle

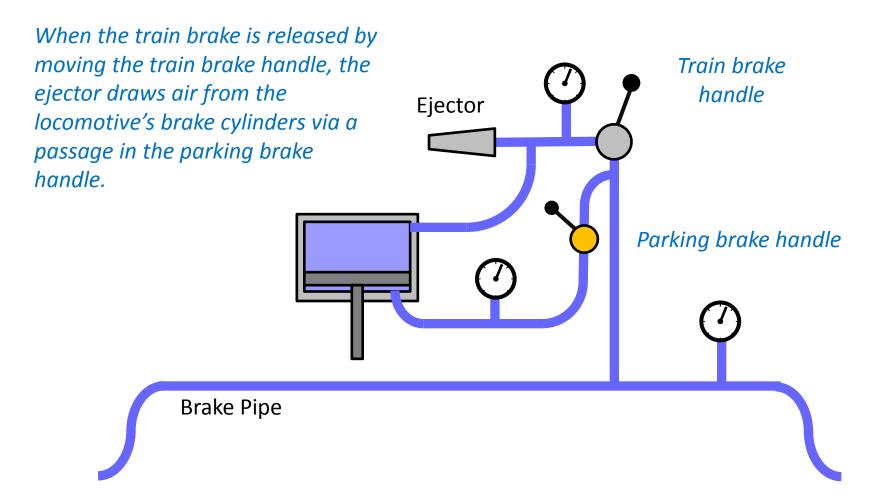
Small ejector handle

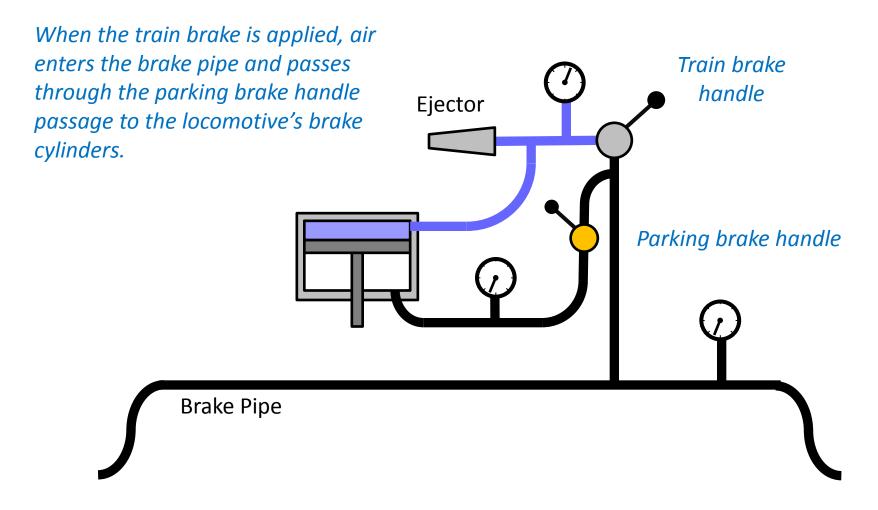
Parking brake handle

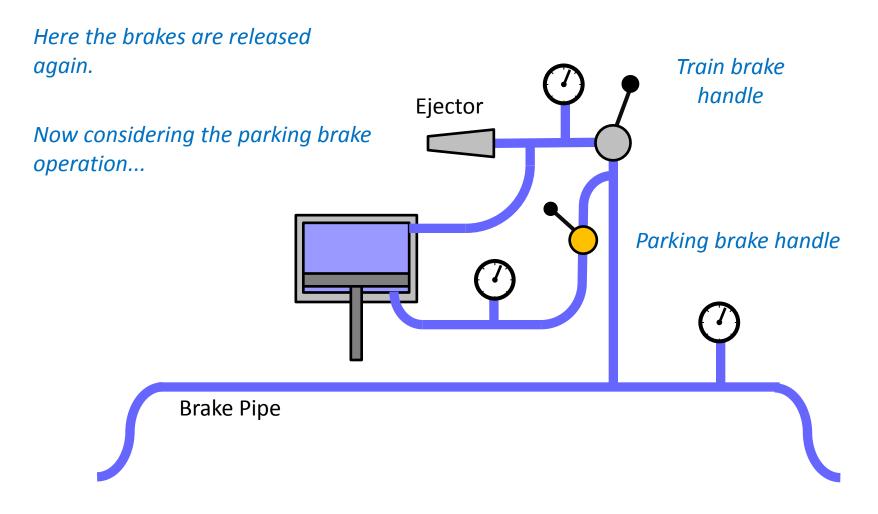


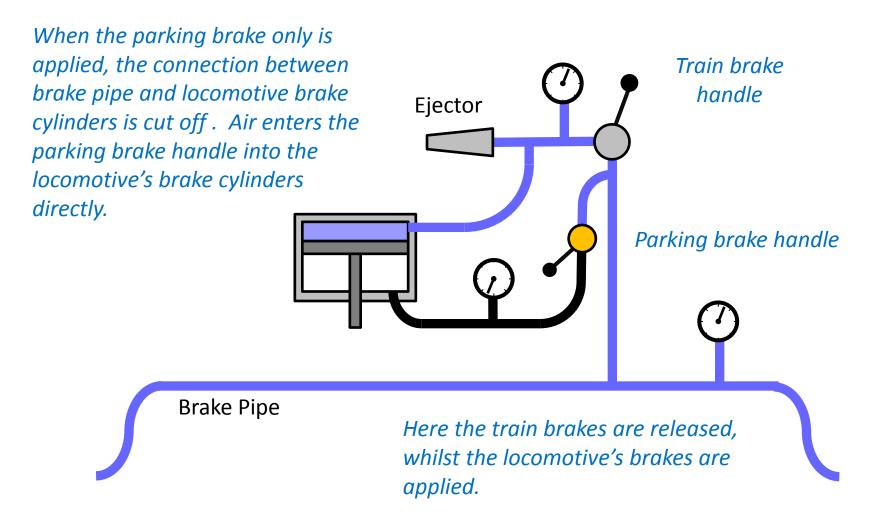












Important notes on the NGG16's parking brake

The locomotive parking brake is ideal for short station stops and waiting at signals. When the train has come to a standstill the parking brake is applied and the train brakes are released. When clear to proceed, the releasing of only the locomotive's brakes will be much quicker than the whole train's brakes.

However, care must be taken if the train is on a gradient, where the parking brake may not be sufficient to hold the train.

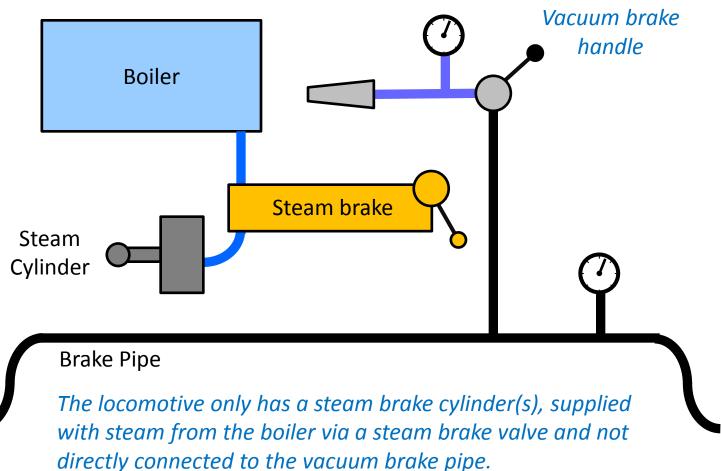
In normal service, the parking brake should not be used to stop a train using only the locomotive's brakes. This could cause excessive wear of the locomotive's brake blocks and sliding of the wheels along the rails. The latter would lead to flats on the driving wheels and subsequent expensive repair.

Important notes on the NGG16's parking brake

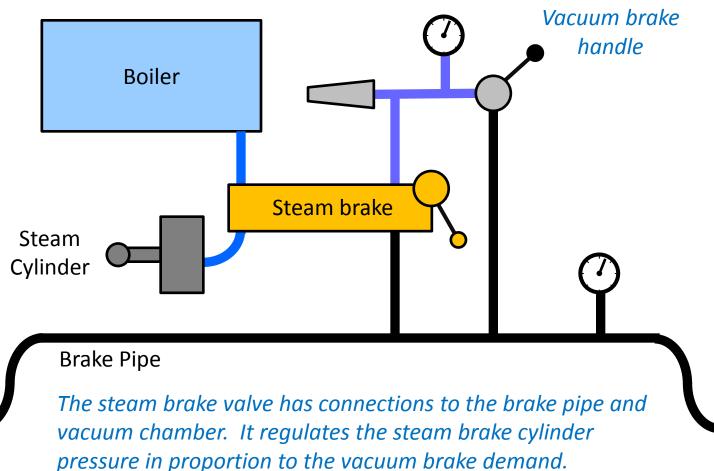
The parking brake controller is not designed to give fine control for braking the locomotive when independent of a train. A light locomotive should still be braked using the train brake controller. The parking brake is intended as a switch, being either on or off.

If referring to the South African Railways (SAR) brake diagrams, be aware that the quick acting valves (intended for use with very long trains) have been removed from the Welsh Highland Railway's locomotives.

Here is part of a diagram of the steam brake.



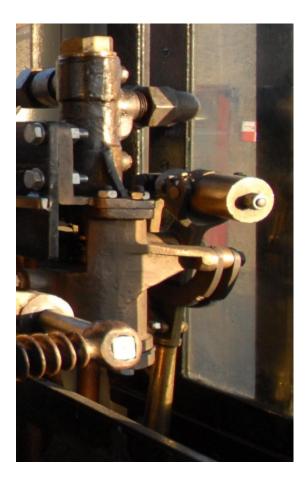
Here is the complete diagram of the steam brake.

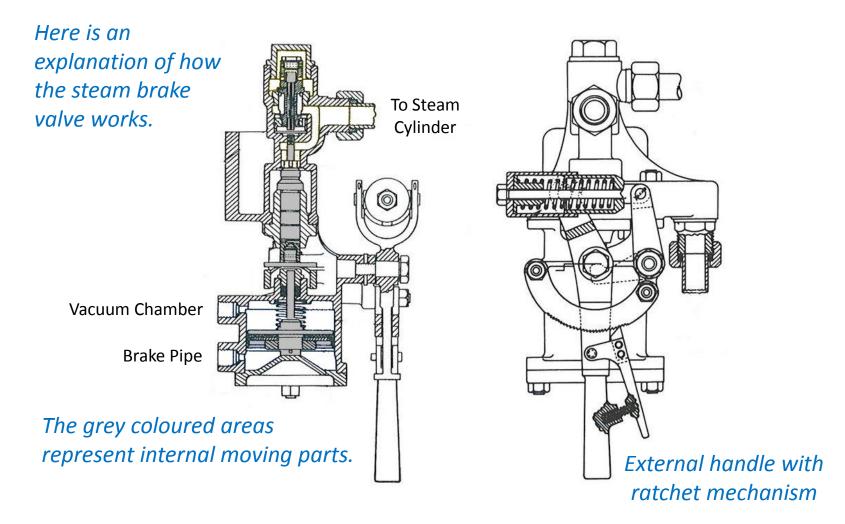


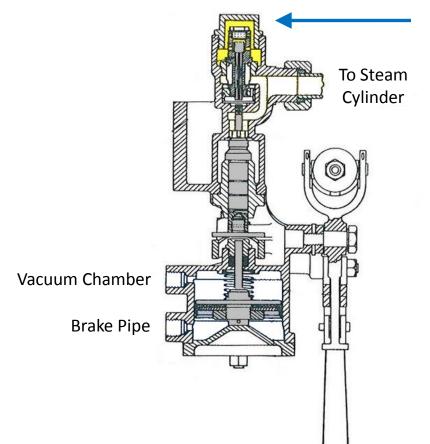
The steam brake valve used as standard on the Ffestiniog and Welsh Highland Railways is a Gresham and Craven design identical to that fitted on the BR standard steam locomotives.

It is mounted in the cab and has an external handle that allows the steam brake to be applied independently of the vacuum brake. It can be used both as a parking brake and when running light loco.

Most locomotives fitted with steam brakes can be run light loco without the vacuum brake system needing to be turned on. However, there are exceptions, for example locomotive number 87. On this locomotive one power bogie requires the vacuum brake.







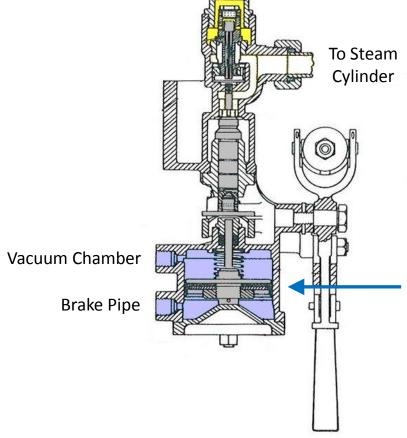
Steam is supplied to the top of the valve. The steam is shown in yellow.

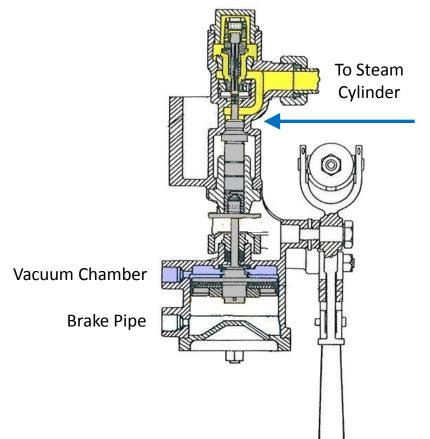
With no vacuum in the system, the steam will not progress any further, because it bears down on the inlet valve, holding it shut.

When the vacuum brake system is turned on, both the brake pipe and vacuum chamber are evacuated of air.

The steam brake valve has a miniature vacuum brake cylinder at its base.

With vacuum in both the brake pipe and vacuum chamber, the miniature piston is in equilibrium and is held at the bottom of the cylinder by a spring.





When a brake application is made, the miniature piston is forced up its cylinder by the incoming air pressure.

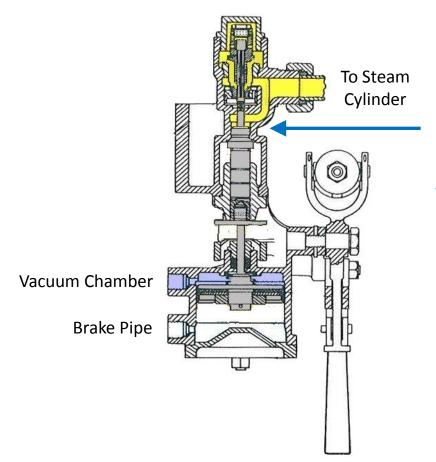
The piston rod pushes up on the steam valve, closing the brake cylinder exhaust outlet and then forcing open the steam inlet.

Steam is supplied from the boiler to the brake cylinder.

The steam pressure entering the brake cylinder bears down on the exhaust valve. This creates a downward force on the piston rod.

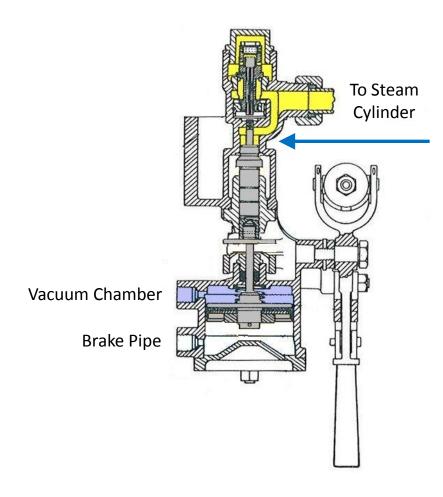
When the steam pressure has become sufficient to balance the input force from the miniature vacuum cylinder, the piston rod moves to allow the steam inlet valve to close. However, the exhaust valve remains shut, so a constant steam pressure is obtained in the brake cylinder.

To Steam Cylinder Vacuum Chamber Brake Pipe



If there is a further reduction in brake pipe vacuum, the miniature vacuum brake cylinder will increase its upwards force on the steam valve. More steam pressure will be allowed to enter the brake cylinder until the forces balance again.

The same event will happen if there is a leak in the steam brake cylinder. If there is a loss of steam pressure, the valve will be forced upwards by the miniature vacuum brake cylinder, allowing more steam to enter the brake cylinder to compensate for the leakage.

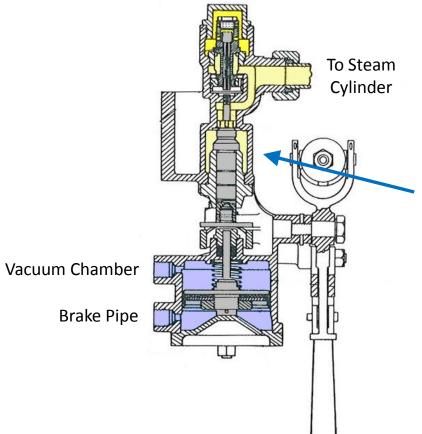


Here the valve has returned to its equilibrium.

The steam in the brake cylinder is being held at a constant pressure in proportion to the vacuum brake demand.

The steam brake valve is known as a proportional valve and it is probably the single most sophisticated valve found on any of the railway's steam locomotives.

Now releasing the brake...



When the brake pipe vacuum is partially or fully restored, there is a reduction in air pressure on the underside of the piston in the miniature brake cylinder.

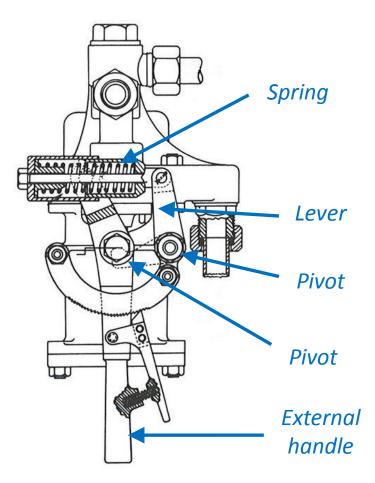
The upwards force on the steam valve is reduced and the steam pressure in the brake cylinder can force open the exhaust passage and start to escape to atmosphere.

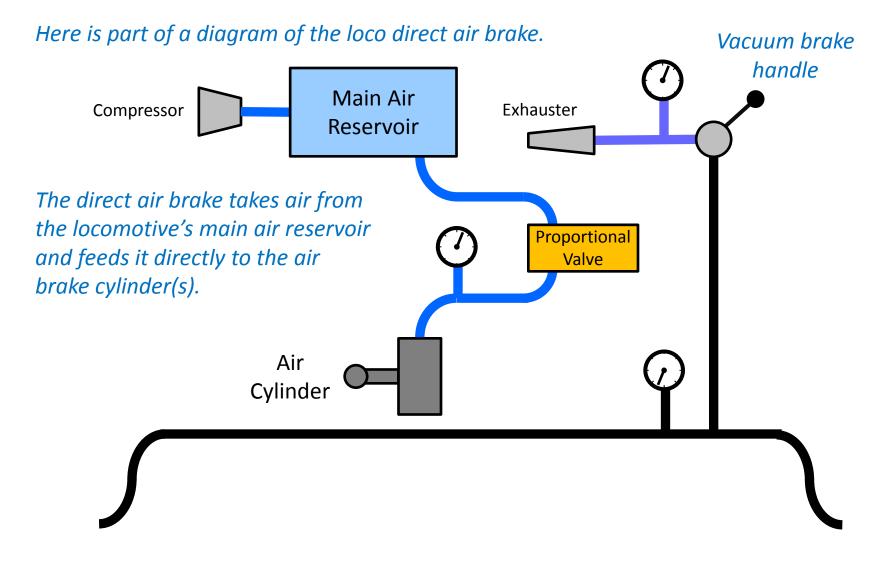
The exhaust of steam will continue until the valve is rebalanced. If vacuum is fully restored, the steam brake will fully release.

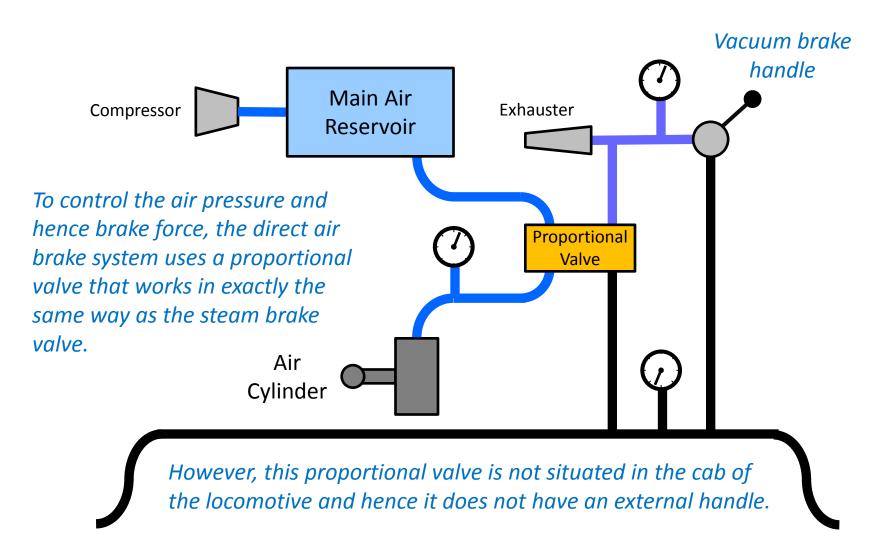
The external handle of the steam brake valve is not directly connected to the internal mechanism. Instead, the handle compresses a spring. This spring then pushes on a separate mechanical lever that is connected to the valve mechanism.

The more the handle is moved, the greater the compression on the spring and the greater the force on the valve mechanism.

In this way the handle takes the place of the internal miniature brake cylinder, applying force to the valve to create a balancing steam pressure.



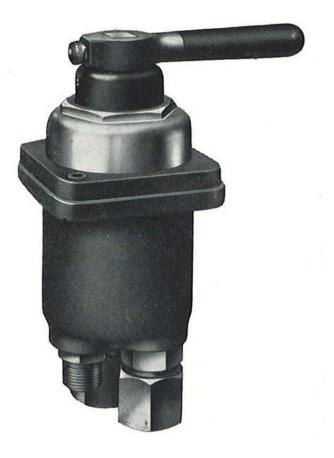


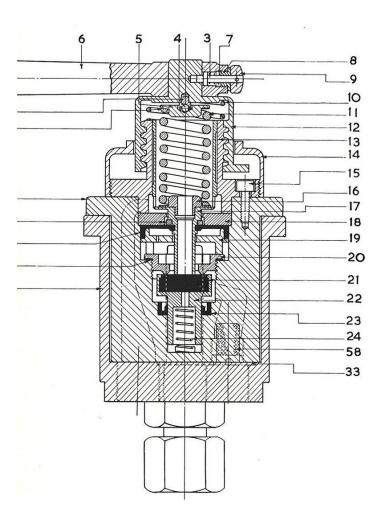


Here is the complete diagram of the loco direct air brake. Vacuum brake handle Main Air Compressor Exhauster Reservoir **Direct Air Brake** Controller Proportional (pressure Valve A separate direct air brake qauges omitted valve is fitted in the cab. **Double Check** for clarity) Valve Air Cylinder

As there are two separate pipes to the brake cylinder, a double check valve is fitted to prevent air from being passed back up the other pipe.

This is a typical direct air brake valve.

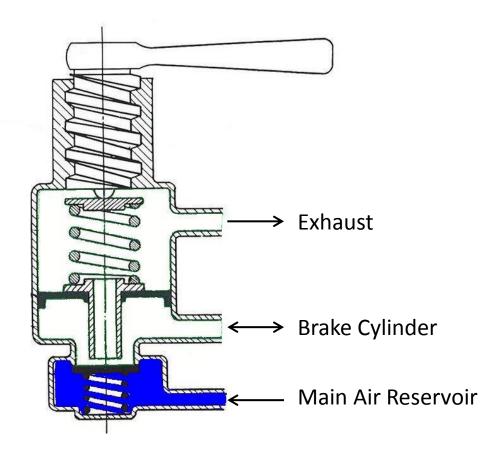




This diagram shows the internal mechanism simplified.

The valve is in the brake released position.

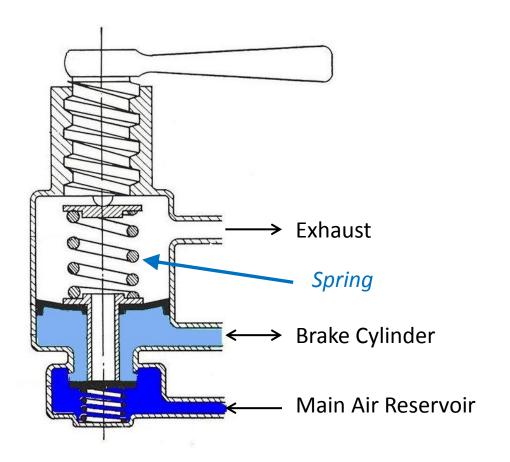
Main air reservoir pressure enters at the base of the valve but cannot proceed any further. It bears up on the inlet valve keeping it firmly shut.



When a brake demand is made by moving the handle, the threaded valve stem causes the internal spring to be compressed.

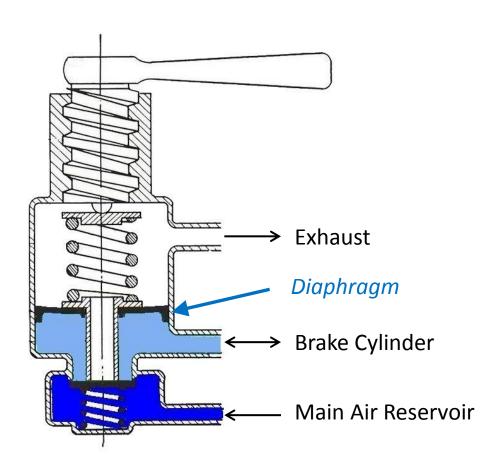
The spring pushes on a hollow valve stem, seating it on the inlet valve.

The inlet valve is then forced open, allowing air pressure to enter the brake cylinder.



The air entering the brake cylinder fills the chamber of the valve that is below the spring. Here it bears on a diaphragm, generating a counter force to the spring.

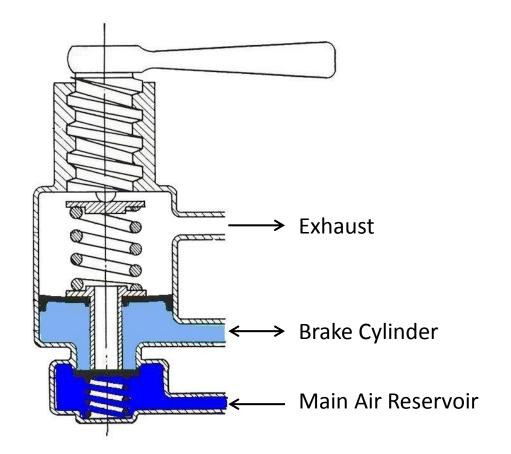
When the air pressure is sufficient to match the spring force, the inlet valve closes and the valve is held in equilibrium.



The air pressure in the brake cylinder is proportional to the distance that the handle has been moved.

If there is a leak of brake cylinder air pressure, the spring will overcome the reduced pressure and will force open the inlet valve to compensate for the leak.

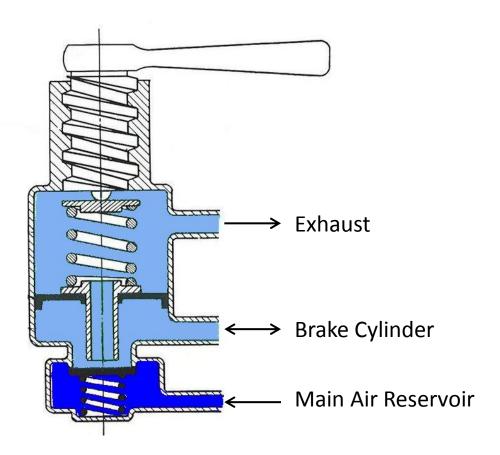
This is known as a <u>self-</u> <u>lapping</u> brake controller.



When the handle is returned towards the release position, the compression of the spring is reduced.

The air pressure in the brake cylinder (and under the diaphragm) overcomes the reduced spring force and lifts the hollow valve stem from the face of the inlet valve.

Air pressure will now escape to atmosphere through the hollow valve stem until the valve is again in equilibrium.



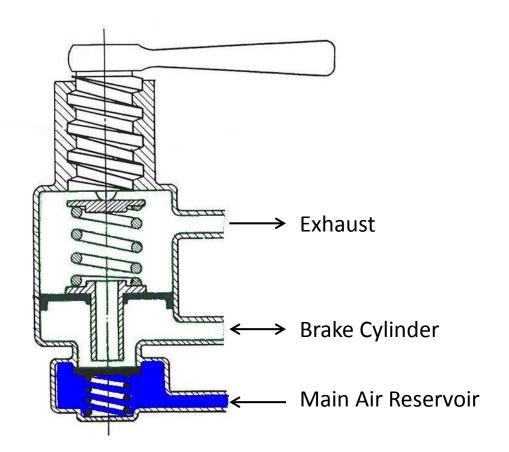
When the handle is returned completely to the release position, there is no compression of the spring.

All the air pressure in the brake cylinder will now escape to atmosphere through the hollow valve stem.

The brake is released.

Question:

What is the drawback with this controller?



Answer:

When compressed, the internal spring will apply a force back on the control handle. This force makes the handle susceptible to unwind into the brake released position when disturbed by any vibration.

Unlike the steam brake valve, there is no ratchet to hold the direct air brake handle in the applied position.

Therefore, a train must not be parked and left unattended with only the locomotive direct air brake applied.

Operation

Question and Answer session

The following slides ask various questions on the operation of the vacuum brake system. Answers providing advice on best practice are based on several years experience of testing brake equipment.

However, this section does not represent any official policies of the Ffestiniog and Welsh Highland Railways. If in any doubt regarding the operation of the brake equipment, contact the Chief Mechanical Engineer at Boston Lodge Works.

Q: How should you carry out the continuity test?

Should you:

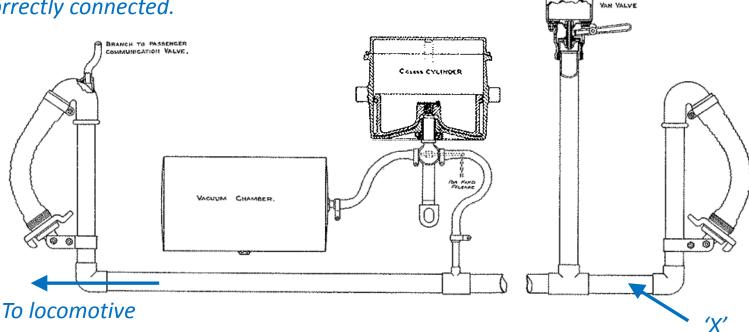
- a) remove the coupling hose at the end of the train and check for air flow into the brake pipe? OR
- b) examine the pressure gauge in the guards compartment?

INGLE GAUGE

Vacuum brakes

Discussion

The pressure gauge in the guard's compartment will not identify a blockage at the position marked 'X'. Nor will it confirm that any vehicles to the right are correctly connected.



Discussion continued

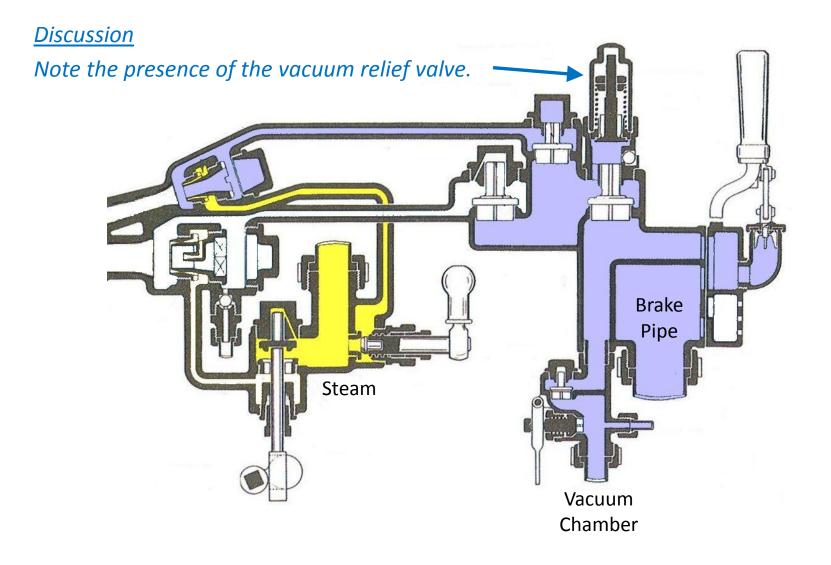
Removing the end brake hose and observing that air enters the brake pipe does not confirm conclusively that there are no blockages. There could be restrictions to the flow of air. The guard's pressure gauge does give confirmation that the required vacuum is being created and destroyed quickly.

In the Republic of Ireland, the brake continuity test is carried out by connecting an end of train device to the last brake hose of the train. This device has both a pressure gauge and a venting cock to allow the operator to ensure correct brake continuity.

- A: In summary, it is best to test for brake continuity at <u>both</u> the end brake hose and the guard's pressure gauge.
- The end brake hose will confirm that the brake pipe is complete.
- The pressure gauge will indicate that the required vacuum is being generated.

Q: How is it best to setup the small ejector?

- Do you set it to just enough to maintain 21 inches of vacuum?
- Why?



Discussion continued

If the ejector tries to generate a vacuum greater than the nominal 21 inches, the vacuum relief valve will open automatically to allow air into the brake pipe to destroy the excess vacuum. Therefore, the energy used to generate the excess vacuum is wasted. This wastes steam from the boiler and ultimately more fuel is burned.

It should be observed that the amount that the steam valve has to be opened to generate and maintain 21 inches of vacuum will depend on the length of the train (and hence the volume of the train brake pipe). Similarly, when running light loco, the least amount of opening will be required of the steam valve to the ejector.

Discussion continued

Care should be taken having run round a train light loco that the small ejector is set to suit the length of the train, other wise dragging brakes will result. This is especially important on the Welsh Highland Railway, where the large and powerful NGG16 Garratt locomotives (weighing 60 tonnes) can easily overcome a brake drag on the train of carriages (weighing around 110 tonnes).

A: In summary:

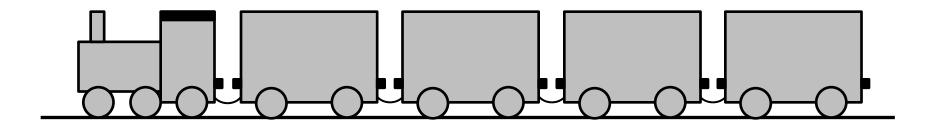
- The small ejector is set at the minimum output required to maintain vacuum in the train pipe.
- However, don't be too mean consider that the NGG16 Garratt locomotives are powerful enough to overcome a brake drag.

Q: What does the term "defensive driving" mean?

• A common perception is that it means accelerating and braking gently, but this is not quite the case...

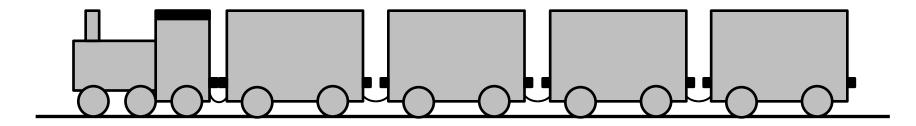
<u> Discussion – Part 1</u>

Consider our train travelling along at constant speed. On level track the locomotive is pulling the train to overcome friction. As a result all the couplings are stretched.



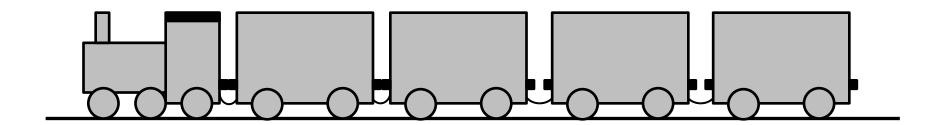
Discussion continued

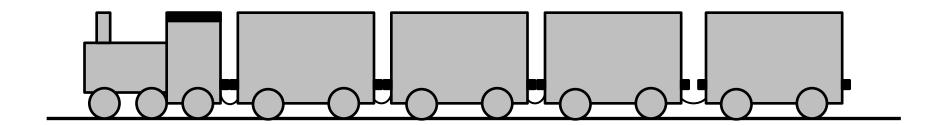
The driver applies the vacuum brake, and air rushes into the brake pipe. As the air enters the brake pipe it will fill the nearest brake cylinders first.



The front of the train will brake before the following vehicles, causing the couplings to be compressed.

The next few slides show this taking effect throughout the train.

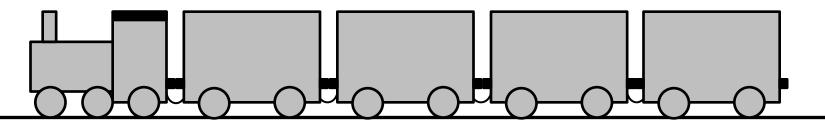




Discussion continued

Once air has filled all the brake cylinders, all the couplings will be compressed.

To help overcome this problem locomotives are traditionally designed to brake less severely than the coaches. Thus the locomotive tends to run away from the train and in doing so tends to stretch the couplings.



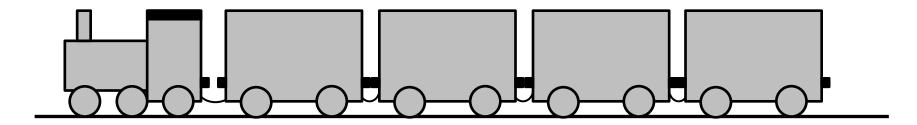
Note that on a unfitted freight train the compressing of the couplings cannot be avoided.

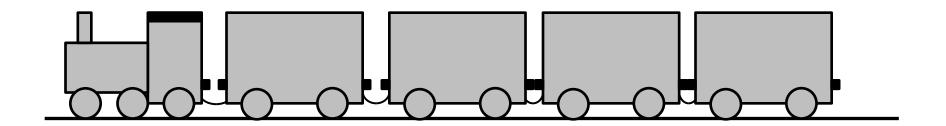
Now we shall consider the brakes being released...

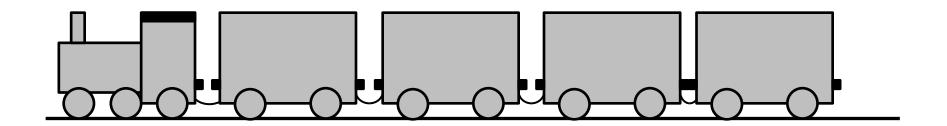
Discussion continued

As the ejector sucks air from the train brake pipe there is a tendency for the brakes on the leading vehicles to release first.

The couplings gradually stretch out, starting at the front of the train...

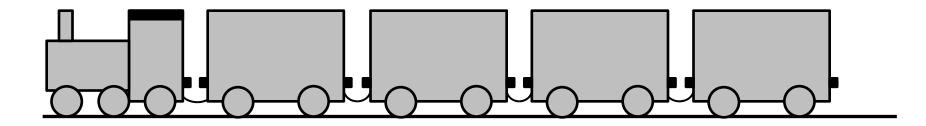






Discussion continued

This is one of the reasons for stopping on a rising vacuum. The train will come to rest with its couplings stretched. When the locomotive begins to haul the train again, there will be less likelihood of jolts to the trailing vehicles.



It is unlikely that stopping on a rising vacuum will stretch all of the train's couplings, but there will be a lot less couplings in which to take up slack when restarting.

Discussion continued

The compression of the couplings results in jolts to the trailing vehicles. The effect will be worse when a greater volume of air is admitted into the brake pipe in a short period of time (i.e. a harsh initial brake application). However, once the train is braking and the couplings are already compressed, increasing the brake effort (by admitting a greater volume of air) will not result in severe jolts to the train.

The same is true for releasing the brakes whilst still moving. The initial release should be gentle (small ejector only) to allow the couplings to stretch gently. Once the couplings are stretched, the brake can be released quickly (using the large ejector) without causing severe jolts. Similarly, the driver should wait until the brakes are all completely released before opening the regulator, otherwise severe jolts will result at any couplings not yet stretched. Initial openings of the regulator should also always be gentle, to stretch the couplings, followed by a greater opening of the regulator (if required).

Discussion continued

Once the train is braking with the couplings compressed, the brake application can be made more severe without causing jolts. However, one very important point must be considered:

Increasing the brake force will increase the deceleration force experienced by the passengers.

The above is fact. The quicker you decelerate the more force you feel.

Discussion continued

Therefore, in summary for normal service brake applications:

Sequence of braking

- 1. Initial Application
- 2. Sustained Braking
- 3. Easing / Release

Severity of braking

- Gentle
- Moderate?
- Gentle

<u> Discussion – Part 2</u>

The best method of braking can be a controversial topic.

Even today, engineers still differ in opinion as to whether braking (over a given distance and from a given speed) should be carried out by one constant gentle application, or by a succession of intermittent harsher brake applications.

The various positives and negatives of each approach are considered on the next slide.

Note that on heritage railways operating at only 25mph maximum speed, the scope for intermittent braking tends to be limited to braking on long gradients to control speed.

Continuous Braking

- Smoother action
- Only one movement of the rigging and brake blocks
- Very light applications can cause brake chatter
- Continuous contact of brake blocks retains heat
- Vacuum must be retained in vacuum chambers

Intermittent Braking

- Can tend to cause jolts
- More movements and impacts of the brake rigging
- Positive and quick motion of blocks onto the wheels
- Allows air to circulate around the brake blocks
- Vacuum chambers are recharged

Discussion continued

From the previous slide, it is clear that the disadvantages of continuous light brake applications tend to be because it is continuous. The disadvantages of intermittent harsher brake applications tend to be because they have to be harsher to achieve the same stopping distance.

However, an intermittent and light brake application omits many of the disadvantages of both of the above. The compromise is that the stopping distance will increase.

To achieve an intermittent and light brake application requires the driver to plan ahead. Similarly, the driver can shut off the regulator and use a rising gradient as a means of slowing the train instead of applying the brake blocks.

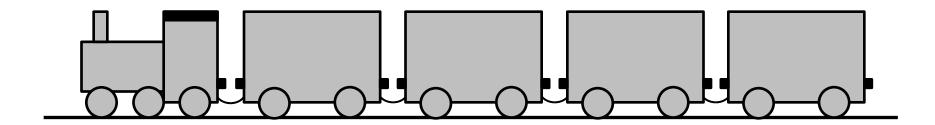
This is, of course, provided that the timetable permits. Although consider how many seconds are actually gained by braking later?

A: The term "defensive driving" means <u>planning ahead</u>.

Q: How would you change your method of braking when ascending or descending a steep gradient?

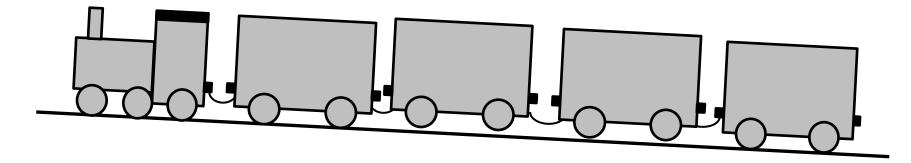
Discussion

Here is a representation of the train on level track.



Discussion continued

Here is a representation of the train ascending a gradient.

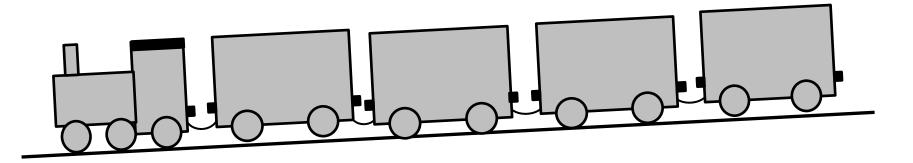


If the gradient is relatively steep and the brake application is relatively gentle, the train's couplings will remain stretched whilst the train is braking.

The gradient will act to assist the braking and in many cases only a light application will be required throughout the duration of braking.

Discussion continued

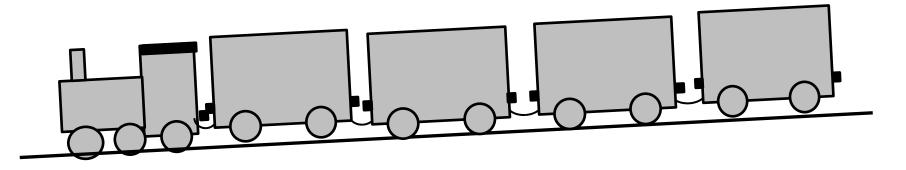
Here is a representation of the train descending a gradient, but not braking.



If the gradient is relatively steep, there will be a tendency for the trailing vehicles to run into the locomotive. This is because the locomotive has more resistances to its free running than trailing vehicles. For example, locomotives tend to have plain bearings, the pistons act like compressors, slowing the rotation of the wheels, and the rigid wheelbase is typically longer than a coach bogie so the curve resistance will also be greater.

Discussion continued

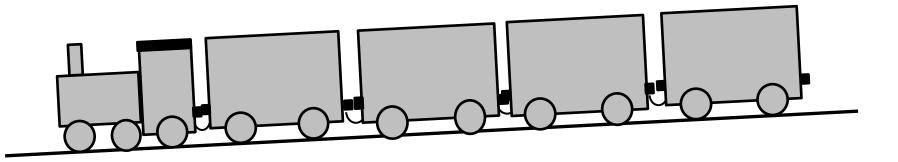
This view shows the trailing vehicles catching up with the locomotive.



Now that the first of the trailing vehicles is resting on the locomotive, it will be slowed down. Hence, the remainder of the trailing vehicles will catch up with the locomotive and first vehicle. This process will continue until all the train's couplings are compressed.

Discussion continued

This view shows the train with couplings compressed.



With most or all of the couplings compressed, a harsher brake application can be initiated sooner without causing severe jolts. This is useful because the steep gradient will try to counter the braking, which therefore requires a harsher application.

continued...

Discussion continued

If the train is going to stop at a station on a steep descending gradient, there is no merit in trying to stretch the couplings by stopping on a rising vacuum. When the train starts again on its descent of the gradient, the vehicles will ultimately tend to run into each other again.

When releasing the brakes on a steep descending gradient with the couplings compressed, there will be a tendency for the brakes to release at the front of the train first (as discussed previously). To minimise this effect the brakes should be released as gently as possible (using the small ejector only and a gradual release).

Note that the previous slides have considered severe gradients. Shallow gradients will not have the same effect and braking in this case should be carried out in the same way as on level track. It may not always be possible to easily distinguish a severe gradient from a shallow one in terms of the compression of the couplings before and during braking.

Discussion continued – Drifting steam

To complicate matters, drifting steam is sometimes used to retain a positive pressure in the locomotive's cylinders. From a train handling point of view, on a steep gradient there are 4 scenarios:

- (1) Drifting steam exceeds the locomotive's resistances and tends to accelerate the train. If countered by braking there will be excess brake block wear. This is not ideal.
- (2) Drifting steam marginally exceeds the locomotive's resistances and keeps the coupling stretched, without increasing speed. This is a good balance.
- (3) Drifting steam reduces locomotive resistance to the same as the trailing vehicles. This results in the vehicles tending to coast into each other as the resistances vary slightly, such as on curves. This is not ideal.
- (4) Drifting steam is not sufficient to overcome the locomotive's resistances, and the couplings remain compressed. The train behaves as described in the previous few slides. This is an acceptable scenario.

Discussion continued – Drifting steam

Whilst scenario (2) is a good position, it is difficult to achieve because it can easily shift to scenarios (1) or (3) as the gradient and curves change.

Scenario (4) is also a good and tends to be easier to achieve. If anyone is in doubt about trains running with their couplings compressed, it is worth noting that the 150 wagon freight trains in the USA brake principally using the locomotive dynamic brake only and not the train brakes.

In any case, judging the changing gradient resistance, curve resistance and drifting steam assistance is not easy.

That single small handle in the cab that controls brake pipe vacuum often looks deceptively as straightforward as the foot pedal brake in your car. Anyone without skill could pull it and cause the train to stop. However, braking a train smoothly for the passengers in trailing vehicles is not easy and should not be underestimated.

A:

On a steep rising gradient, it should be possible to apply the brakes gently enough that the couplings remain stretched.

On a steep falling gradient, where the couplings tend to be compressed, the brakes should be released gently.

Braking smoothly is not easy.

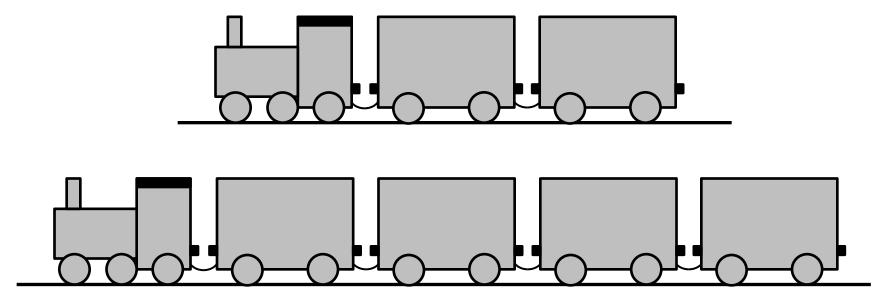
Q: How is it best to determine the severity of the brake application?

Is it:

- a) By experience?
- b) By judgement?
- c) By referring to the pressure gauge?

Discussion

These drawings provide a clue. Consider how the brakes will behave differently for the different lengths of train.



Discussion continued

If you double the length of the train you double the volume of air required to achieve the same severity of brake application. This is because the brake pipe is twice as long and there are double the number of brake cylinders to fill with air.

The brake controller will have to be applied for longer to allow sufficient time for the greater amount of air to enter the brake pipe.

A driver familiar with short train formations will tend to under-brake a long train. A driver familiar with long train formations will tend to over-brake a short train.

Discussion continued

This is not to detract too much from experience. A certain amount of experience is also beneficial. The pressure gauges in the locomotive cab only illustrate the pressure at that point in the train. Changes in pressure can take several seconds to take effect throughout the train. Pressure gauges can also be faulty.

It is important to remember that whilst the pressure gauge informs you of the brake demand being made along the brake pipe, it does not represent an actual deceleration of the train. The latter could be affected by slippery rails. Judgement of the weather conditions and reference to other gauges like the speedometer are also important.

A: Best practice is to use a combination of experience, judgement and the pressure gauge.

Don't ignore the pressure gauge. It is there for a reason.

Enhancements

<u>Enhancements</u>

The following slides look at equipment that can be fitted onto vacuum brake systems to improve their performance:

- Vacuum Pump *
- Piston head non-return valve
- Slipping band piston seal *
- Direct admission valves *

As of 2011, the items marked * are not fitted on Ffestiniog and Welsh Highland Railway rolling stock. However, it is beneficial to be aware of them because it is quite possible they could be fitted in future. Also, if you visit any other railways as guest drivers it is beneficial to be aware that their vacuum brake systems may behave differently as a result of such enhancements.

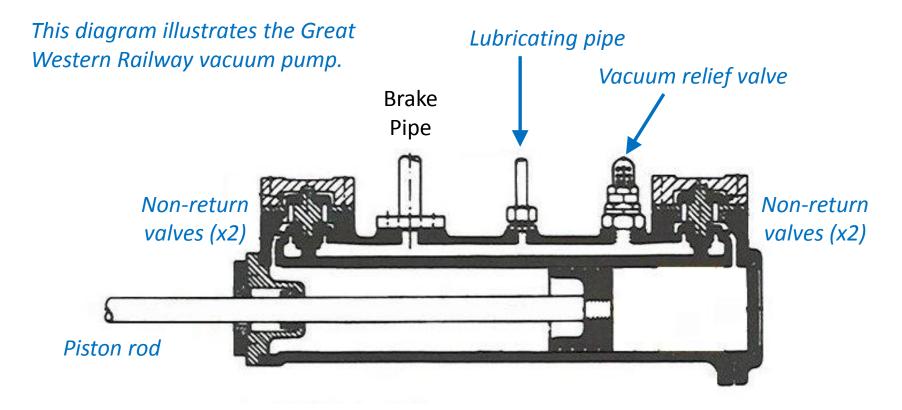
Vacuum Pump

The vacuum pump is a device which is directly connected to the locomotive's motion. The pump operates whenever the locomotive is moving, and draws air from the brake pipe so that the ejector can be turned off, conserving steam.

When the regulator is open, the vacuum pump is of little benefit. A portion of the steam being supplied to the cylinders is being used to overcome the resistance of the pump.

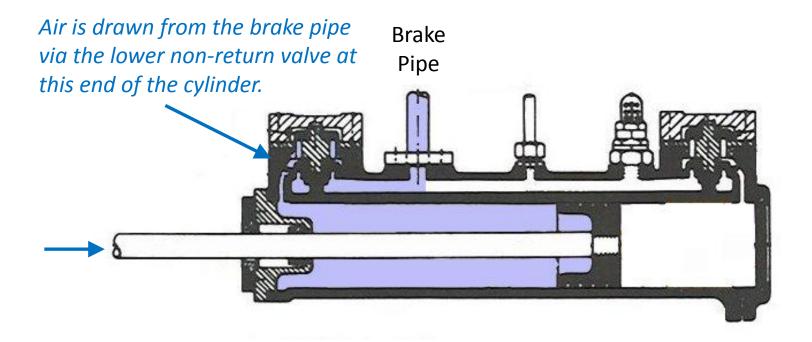
However, when the regulator is closed the pump maintains brake pipe vacuum without using any steam. Also, the pump's resistance provides brake effort without brake block wear. The vacuum pump conserves steam usage and therefore saves fuel.

On the Ffestiniog Railway's 11 miles of continuous downhill running, a vacuum pump would offer potentially large savings in steam and fuel usage.

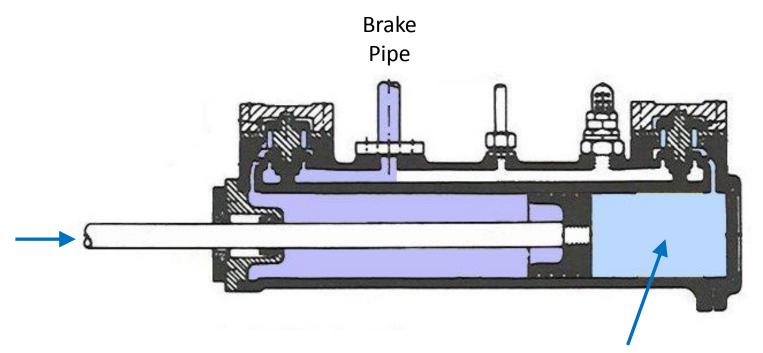


The vacuum pump is fitted in close proximity to one of the locomotive's steam cylinders. The piston rod of the vacuum pump is connected to the crosshead, so that the pump is moved back and forth whenever the locomotive is moving.

In this diagram the piston rod is being pushed to the right.

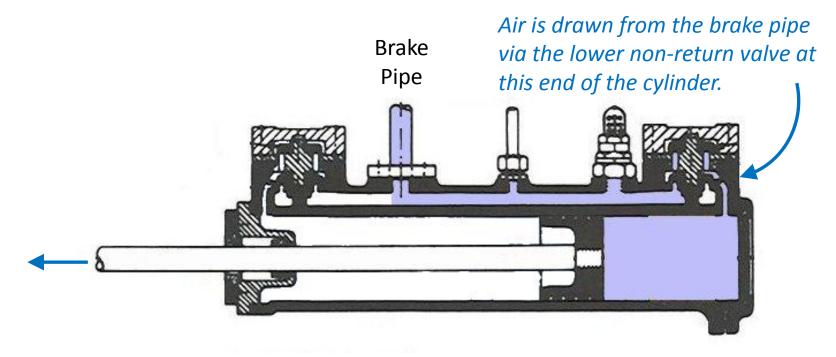


In this diagram the piston rod is being pushed to the right.

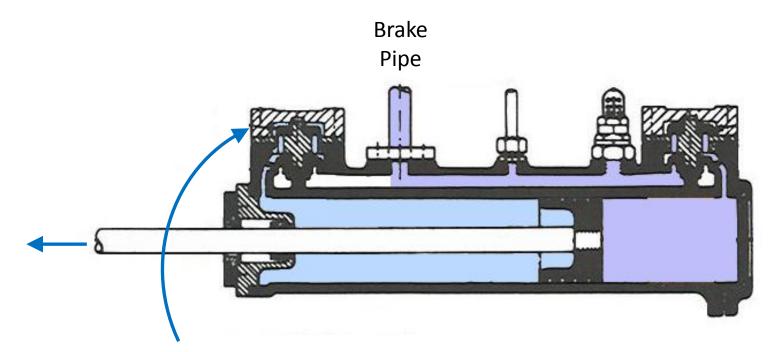


Air would be compressed at this end of the cylinder if it cannot escape. This is why there are two non-return valves. The upper non-return valve at this end of the cylinder allows the air to escape to the surrounding atmosphere.

In this diagram the piston rod is being pulled to the left.



In this diagram the piston rod is being pulled to the left.



The upper non-return value at this end of the cylinder allows the air drawn from the brake pipe on the previous stroke of the piston to now escape to the surrounding atmosphere.

Ejector

In the next few diagrams we will consider how the vacuum pump can be fitted into the brake system.

This diagram shows the standard system on a locomotive <u>without</u> a vacuum pump.

Brake Pipe

Ejector

Pump

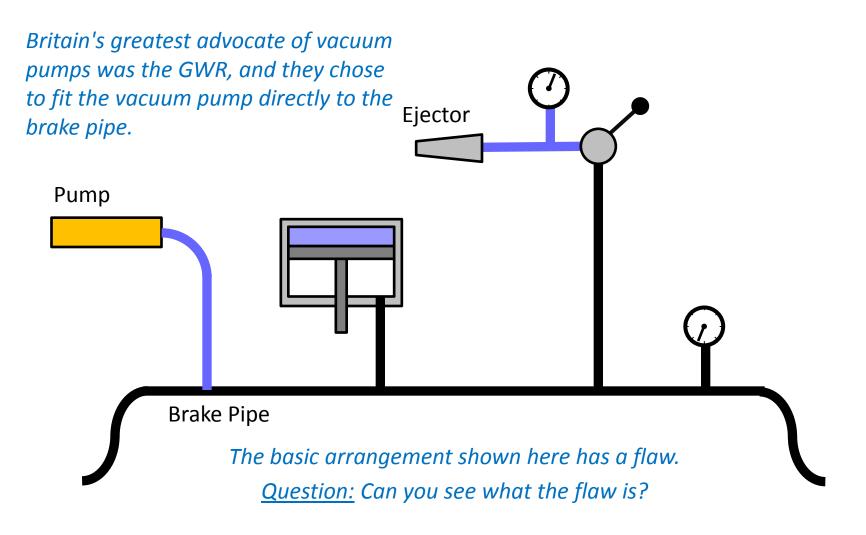
A straightforward option is to fit the pump in parallel to the ejector. Each would have a non-return valve to prevent air being drawn back in when it is not in use.

Brake Pipe

The vacuum pump can complicate the operation of the brakes, because it behaves differently to a steam powered ejector. An ejector provides a constant suck at all times, whereas a vacuum pump provides a greater suck when the locomotive is moving faster.

Therefore, the feel of the continuous type of controller (i.e. no lap position) would be different when the train is moving faster (when the suck is greater). At slow speeds there may be very little suck and operation of the ejector could be required to release the brakes. At high speeds the controller may have to be operated harshly (a greater movement of the handle) to allow the incoming air to beat the suck of the pump.

The arrangement illustrated on the previous slide would work better with a controller comprising a lap position. Such a controller would physically disconnect both the ejector and pump from the brake pipe during braking. However, the operation of the controller and the ejector could get confusing.



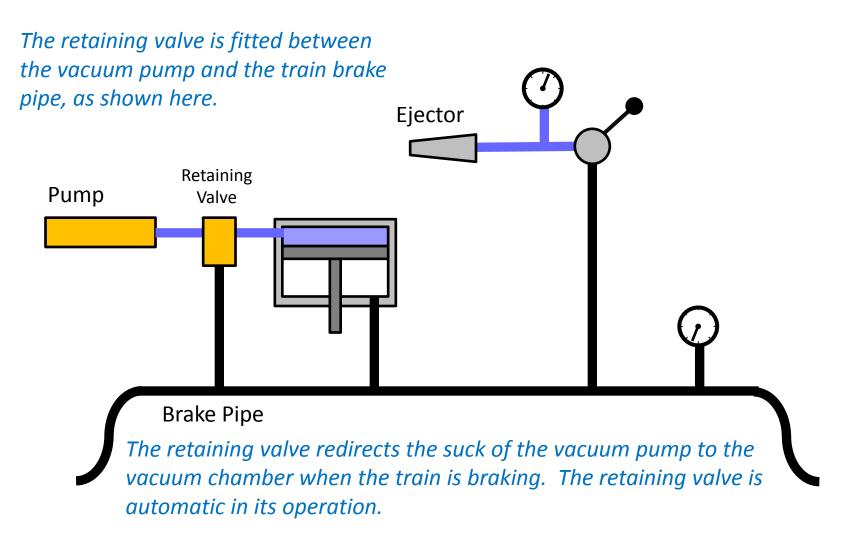
Answer:

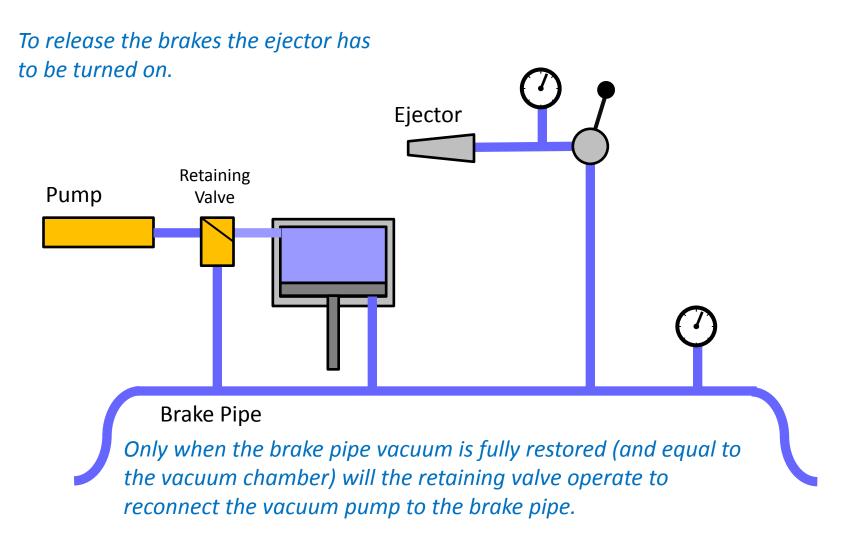
The vacuum pump will fight the brake application made by the driver's control valve. The effect of this will be worse the faster the train is moving.

As the train moves faster, the vacuum pump moves faster and pumps a greater volume of air from the train brake pipe.

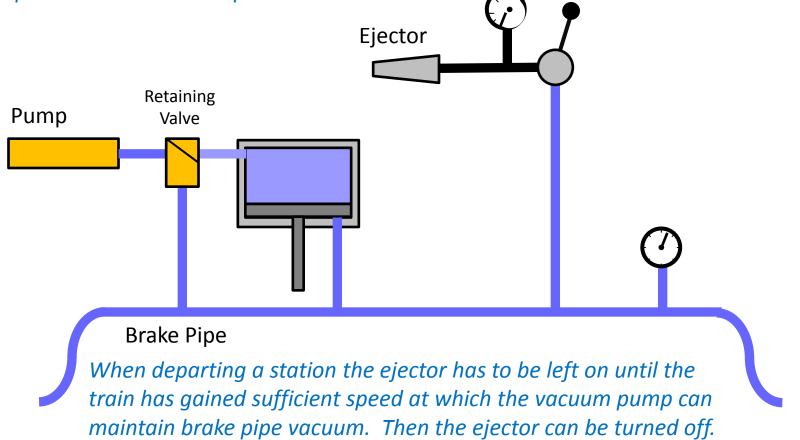
This would create an additional difficulty for the driver, who would have to operate the brake control more severely at higher speeds. Also, if double heading, there would be two vacuum pumps fighting the brake application, further complicating the whole situation.

Therefore, the GWR fitted an additional component, known as the retaining valve...





The suck of the vacuum pump is proportional to the train speed.



Vacuum Pump – Operation

The operation of the vacuum brake controller on former GWR locomotives is different to convention because of the retaining valve. When the controller is moved to make a brake application, the vacuum pump becomes isolated indefinitely. This means that if the controller is held in the application position the brake pressure will eventually build up to maximum.

To hold a partial brake application, the controller must be returned to the running position, which is now a lap position. Having made the initial reduction of brake pipe vacuum, no air will be drawn from the brake pipe.

To increase the braking force, the controller would be momentarily moved to let more air in and then returned to the running (lap) position. To ease the brake, the ejector has to be turned on to suck air from the brake pipe.

Vacuum Pump – Operation continued

During shunting operations, the ejector is typically left running. This is because the slow speed of shunting operations is insufficient for the vacuum pump to maintain the vacuum.

Depending on the type of locomotive, the GWR had two different arrangements. Some locomotives had separate ejector and brake control handles. Other locomotives were arranged with one control handle, which included ejector, lap and brake application positions.

On locomotives fitted with separate ejector and brake control handles, when the ejector is running continuously the brake controller will operate more conventionally like a dreadnought. The more the handle is pulled towards the apply position, the greater the severity of the brake (i.e. there is no lap position).

Vacuum Pump – Hazards

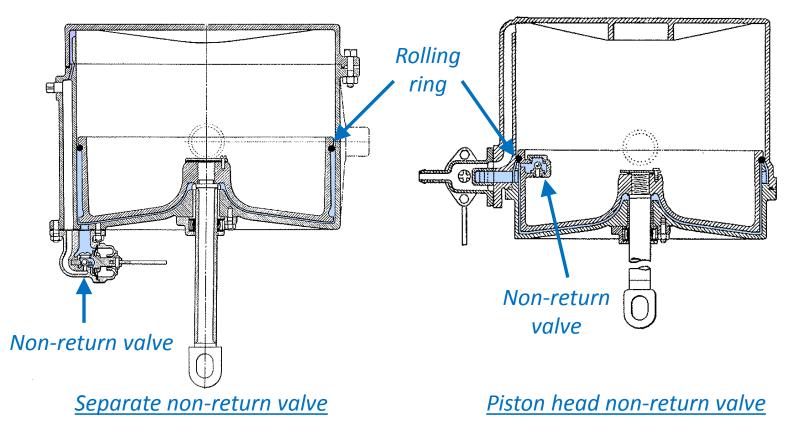
The retaining value only operates when it senses the initial inrush of air, and only returns when the brake pipe vacuum is fully restored.

There is a myth that the retaining valve will prevent a strong vacuum pump (when operating quickly) from preventing a brake application. This is not the case. The brake controller fitted to the locomotive must be able of admitting enough air into the brake pipe to partially destroy the vacuum sufficiently for the retaining valve to operate.

An interesting scenario would be the coupling together of a train of locomotives, each with vacuum pump, and trying to make a brake application whilst all these pumps are drawing air from the brake pipe. It would not be surprising if the GWR had specified reduced speed limits when hauling locomotives!

Piston Head Non-Return Valve

These diagrams are explained on the next slide.



Piston Head Non-Return Valve

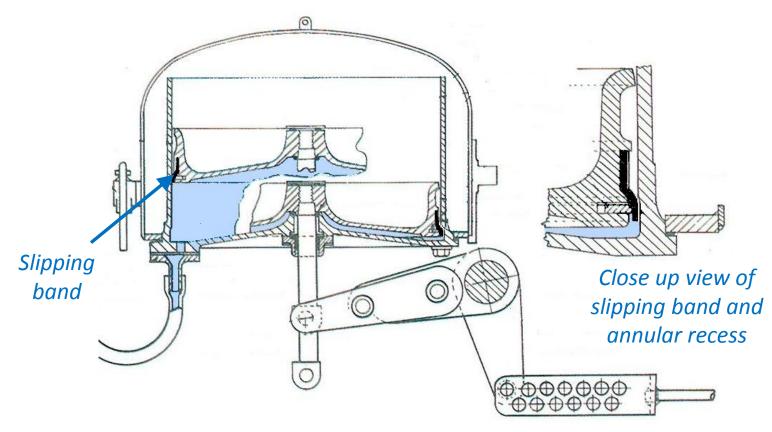
When a brake application is made, the piston is forced up the cylinder and there is a pressure difference between the brake pipe and vacuum chamber. The higher pressure of the brake pipe will try to leak into the vacuum chamber and the pressures gradually equalise. Eventually the brake force will be lost. The better the seal between the brake pipe and vacuum chamber, the longer the brake is maintained.

In the brake cylinder shown on the left, the non-return value is mounted on an external bracket. When a brake application is made there are two areas for a leak to occur. One is the rolling rubber ring seal around the piston and the other is the metal non-return value.

In the brake cylinder on the right, the non-return valve passes to the upper side of the rolling ring when a brake application is made. Therefore the non-return valve is isolated from the brake pipe and the only leak source is the rolling rubber ring.

Slipping Band Piston Seal

This diagram is explained on the next slide.



Slipping Band Piston Seal

This type of seal is arranged so the higher pressure will tend to push the band outwards against the cylinder walls and thus improve the seal.

If air pressure is greater on the upper side of the piston it will tend to push the band inwards and force its way around the outside. In effect, the band forms a non-return valve. To assist this process, there is an annular recess in the cylinder that allows excess air to pass the slipping band when the piston is at the bottom of the cylinder (i.e. the brakes released position).

There is no separate non-return valve required with this type of brake cylinder.

It was used by the Great Western Railway and in many respects is similar to the seal used in air brake cylinders.

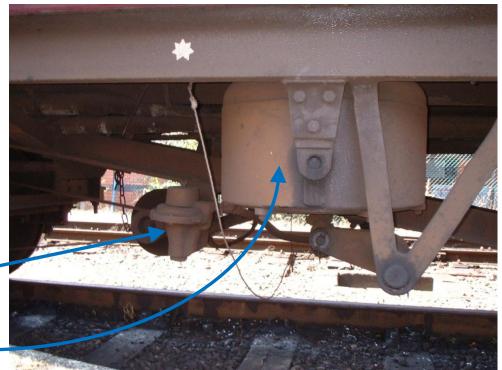
Direct Application Valve

The previous section (operation questions and answers) highlighted how many problems are caused by the slow propagation of atmospheric air along the long brake pipe.

These problems can be largely overcome by the fitting of direct application valves. These valves are mounted next to the vacuum brake cylinders.

Direct Application Valve

Vacuum Brake Cylinder



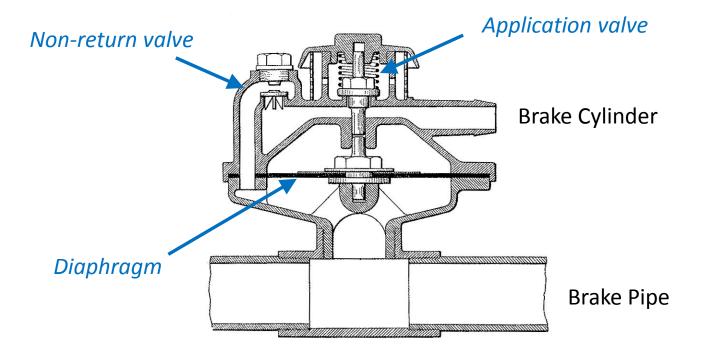
Direct Application Valve

The direct application value operates automatically when the brake pipe pressure increases. It admits air to the brake cylinder directly from the surrounding atmosphere. This means that the air pressure in the brake pipe is not required to fill that particular cylinder and can pass straight on to the next vehicle of the train.

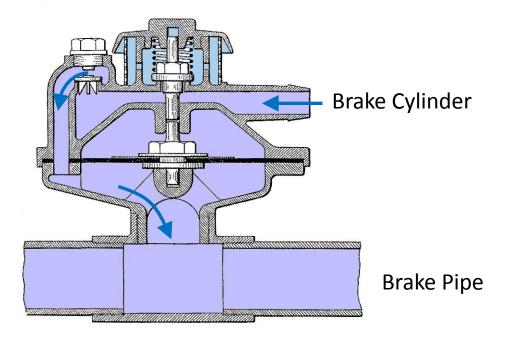
The direct application valve uses the natural reservoir of air around the train to fill the brake cylinders, rather than all the air having to enter at the driver's control valve. The result is faster propagation of the brake demand along the length of the train. The more direct application valves are fitted in a train, the quicker the brake propagation.

<u>Note:</u> easing of the brake will not be speeded up, all the air still has to be drawn out of the brake pipe by the locomotive's ejector.

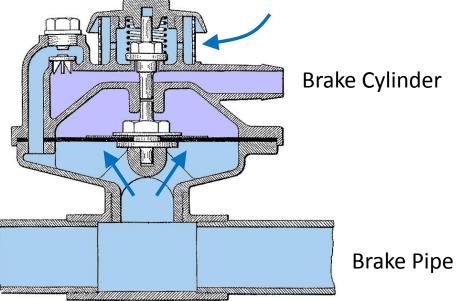
Here follows an explanation of how one type of direct application valve works.



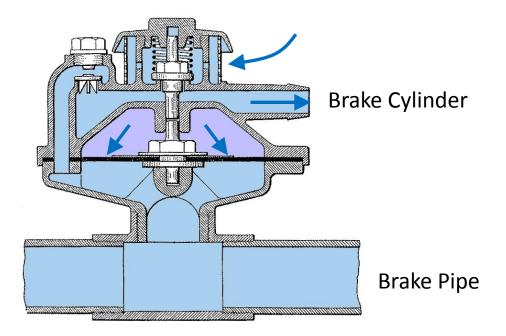
When air is drawn out of the brake pipe, it is also drawn past the non-return valve from the brake cylinder, as shown below:



When air enters the brake pipe, it cannot pass through the non-return valve. The pressure builds up on the underside of the diaphragm and forces open the application valve, allowing air from the surrounding atmosphere to enter the brake cylinder:



As air enters the brake cylinder, it seeps into the chamber above the diaphragm, destroying the vacuum in this chamber:



When the pressures above and below the diaphragm are equal, the spring closes the application valve. The air pressure admitted to the brake cylinder is equal to the pressure in the brake pipe.

